

CM3120: Module 2

Unsteady State Heat Transfer

- I. Introduction
- II. Unsteady Microscopic Energy Balance—(slash and burn)
- III. Unsteady Macroscopic Energy Balance
- IV. Dimensional Analysis (unsteady)—Biot number, Fourier number
- V. Low Biot number solutions—Lumped parameter analysis
- VI. Short Cut Solutions—(initial temperature T_0 ; finite h), Gurney and Lurie charts (as a function of position, $m = \frac{1}{Bi}$, and Fo); Heissler charts (center point only, as a function of $m = 1/Bi$, and Fo)
- VII. Full Analytical Solutions (stretch)

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CM3120: Module 2

Module 2 Lecture IV
Dimensional Analysis
For Unsteady State Heat Transfer



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www.chem.mtu.edu/~fmorriso/cm3120/cm3120.html

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We used **microscopic** unsteady state energy balance modeling to solve one practical problem.

Solution Summary:

Example: When will my pipes freeze?
 The temperature has been 30°F for a while and is supposed to fall to the ground to this temperature for some time of both below the surface. Suggests the temperature drops to 20°F. Modeling will be done for freezing temperatures. (DWT is reaching pipes, which are 8 ft under ground?)

Answer:
 $t = 480 \text{ hours} \approx 20 \text{ days}$

Parameters: $h = 20 \text{ (im)}$, $\alpha = 0.001 \text{ (im}^2/\text{s)}$, $k = 0.5 \text{ (im}^2/\text{F} \cdot \text{s)}$

We used **macroscopic** unsteady state energy balance modeling to solve another practical problem.

CM3120 Module 2—Cooling of a recently manufactured part

Example: Brass parts (oddy shaped, mass M with surface area S) are ejected at regular intervals from a machine that fabricates them. When ejected, the very hot parts at temperature T_0 enter a moving air stream where the air temperature is T_{bulk} . Create a model that will allow us to calculate the temperature of the part as a function of time. Using Excel, calculate $T(t)$ for the parts.

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How can we organize our tool belt?

What is our usual strategy for complex phenomena?

Answer: Dimensional Analysis

Heat Transfer: Steady vs. Unsteady

What are the various cases that are seen?

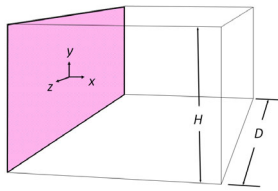
- If h_1 is large, the wall temp is just the bulk temp (fast convection)
- If k is large, the temp profile is straight (quasi-steady state in the slab) and the convection works to keep up (heat xfer limited by h_1 : fast conduction in slab)
- If neither mechanism dominates, it's complicated!

- ✓ Let's nondimensionalize the governing equations and BCs.
- ✓ Let's sort out the various unsteady cases.

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1D Heat Transfer: Unsteady State

**Let's nondimensionalize the governing equations and BCs.
Let's sort out the various cases.**



1D Heat Transfer: Unsteady State

Unsteady State Heat Conduction in a Semi-Infinite Slab

$$\frac{\partial T}{\partial t} = \frac{k}{\rho \hat{c}_p} \left(\frac{\partial^2 T}{\partial x^2} \right) = \alpha \left(\frac{\partial^2 T}{\partial x^2} \right)$$

thermal diffusivity $\alpha \equiv \frac{k}{\rho \hat{c}_p}$

Initial condition: $t = 0 \quad T = T_0 \quad \forall x$

Boundary conditions:

$$x = 0 \quad \frac{q_x}{A} = -k \frac{dT}{dx} = h(T_1 - T) \quad t > 0$$

$$x = \infty \quad T = T_0 \quad \forall t$$

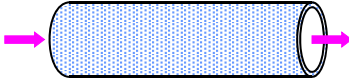
(Review:
How did we do this before?)

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Method:

- Identify the governing equation(s)
- Choose "typical" values (scale factors)
- Use them to scale the equations

We'll modify our solution for **Forced Convection Heat Transfer**



CM3110
REVIEW

Pipe flow

Dimensional Analysis

non-dimensional variables:

time: $t^* \equiv \frac{tV}{D}$	position: $r^* \equiv \frac{r}{D}$ $z^* \equiv \frac{z}{D}$	velocity: $v_z^* \equiv \frac{v_z}{V}$ $v_r^* \equiv \frac{v_r}{V}$ $v_\theta^* \equiv \frac{v_\theta}{V}$	driving force: $P^* \equiv \frac{P}{\rho V^2}$ $g_z^* \equiv \frac{g_z}{g}$
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Energy

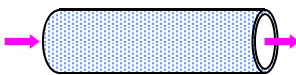
non-dimensional variables:

position: $r^* \equiv \frac{r}{D}$ $z^* \equiv \frac{z}{D}$	temperature: $T^* \equiv \frac{T - T_0}{(T_1 - T_0)}$	source: $S^* \equiv \frac{S}{S_0}$
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**CM3110
REVIEW**

We'll modify our solution for
Forced Convective Heat Transfer



Pipe flow

Dimensional Analysis

non-dimensional variables:

time: $t^* \equiv \frac{tV}{D}$	position: $r^* \equiv \frac{r}{D}$ $z^* \equiv \frac{z}{D}$	velocity: $v_z^* \equiv \frac{v_z}{V}$ $v_r^* \equiv \frac{v_r}{V}$ $v_\theta^* \equiv \frac{v_\theta}{V}$	driving force: $P^* \equiv \frac{P}{\rho V^2}$ $g_z^* \equiv \frac{g_z}{g}$
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$t_{char} = \frac{D}{V}$
 (forced convection)

Energy

non-dimensional variables:

position: $r^* \equiv \frac{r}{D}$ $z^* \equiv \frac{z}{D}$	temperature: $T^* \equiv \frac{T - T_o}{(T_1 - T_o)}$	source: $S^* \equiv \frac{S}{S_o}$
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Slight problem: We need to nondimensionalize t for the unsteady case also, but there is **no characteristic velocity** in thermal conduction in a solid.

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Choice:

For the unsteady case we'll choose a characteristic time based on the thermal diffusivity, α .

$$\frac{t}{t_{char}} = t^* \equiv \frac{\alpha t}{D^2_{char}}$$

This dimensionless time is called Fourier number Fo.

$$t_{char} = \frac{D^2}{\alpha} = \frac{D^2 \rho \hat{c}_p}{k}$$
 (thermal diffusion)

1D Heat Transfer: Unsteady State

Unsteady State Heat Conduction in a Semi-Infinite Slab

$$\frac{\partial T}{\partial t} = \frac{k}{\rho \hat{c}_p} \left(\frac{\partial^2 T}{\partial x^2} \right) = \alpha \left(\frac{\partial^2 T}{\partial x^2} \right)$$

thermal diffusivity
 $\alpha \equiv \frac{k}{\rho \hat{c}_p}$

Initial condition: $t = 0 \quad T = T_0 \quad \forall x$

Boundary conditions:

$x = 0 \quad \frac{q_x}{A} = -k \frac{dT}{dx} = h(T_1 - T) \quad t > 0$

$x = \infty \quad T = T_0 \quad \forall t$

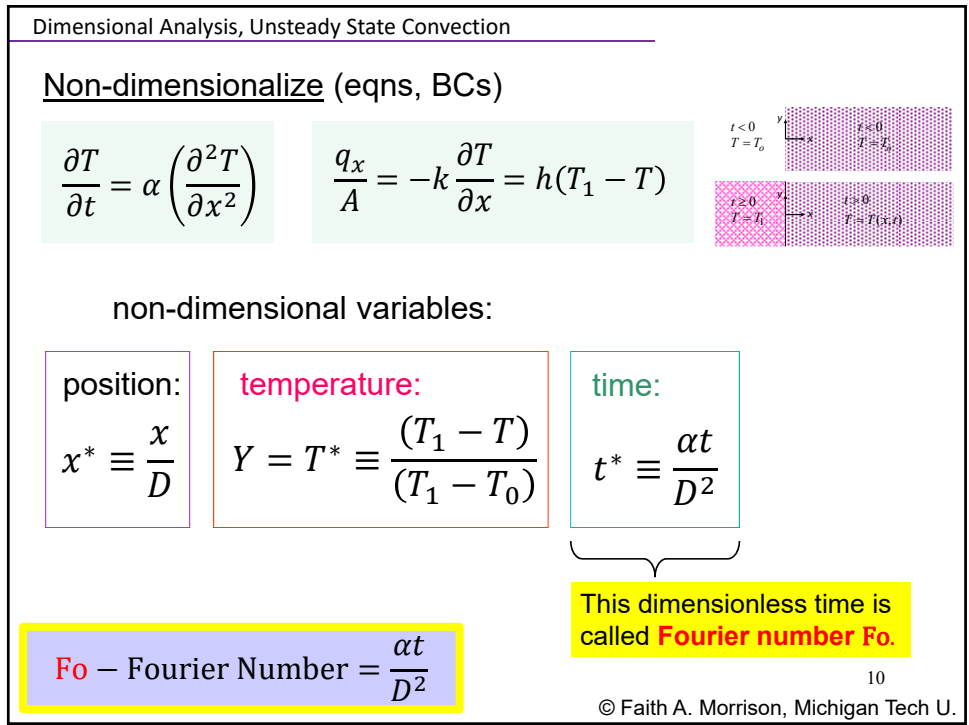
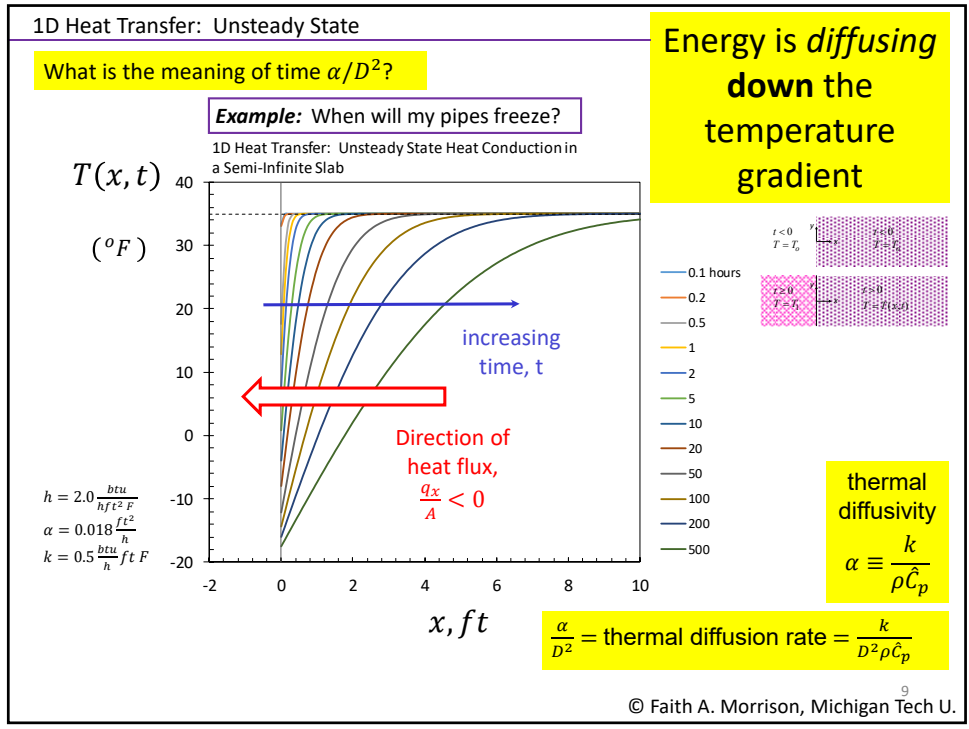
thermal diffusivity

$\alpha \equiv \frac{k}{\rho \hat{c}_p}$

(Appears in the energy balance)

$\frac{D^2}{\alpha} = \text{thermal diffusion time} = \frac{D^2 \rho \hat{c}_p}{k}$

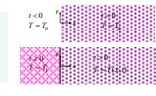
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1D Heat Transfer: Unsteady State

Dimensional Analysis, Unsteady State Convection

Non-dimensionalize (eqns, BCs)

$$\frac{\partial T}{\partial t} = \alpha \left(\frac{\partial^2 T}{\partial x^2} \right) \quad q_x = -k \frac{\partial T}{\partial x} = hA(T_1 - T)$$


non-dimensional variables:

position: $x^* \equiv \frac{x}{D}$

temperature: $Y \equiv \frac{(T_1 - T)}{(T_1 - T_0)}$

time: $t^* \equiv \frac{\alpha t}{D^2}$

Fo – Fourier Number = $\frac{\alpha t}{D^2}$

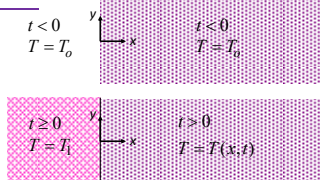
This dimensionless time is called Fourier number Fo.

Let's do it.

11
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1D Heat Transfer: Unsteady State

Unsteady State Heat Conduction in a Semi-Infinite Slab



$$\frac{\partial Y}{\partial t^*} = \frac{\partial^2 Y}{\partial x^{*2}}$$

temperature: $Y \equiv \frac{(T_1 - T)}{(T_1 - T_0)}$

Initial condition: $t^* = 0 \quad Y = 1 \quad \forall x^*$

Boundary conditions: $x^* = \infty \quad Y = 1 \quad \forall t^*$

$x^* = 0 \quad \frac{\partial Y}{\partial x^*} = \text{Bi} Y \quad t^* > 0$

$\text{Bi} \equiv \frac{hD}{k}$

Bi – Biot Number = $\frac{hD}{k}$

12
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In dimensionless form, we see that this problem reduces to

$$Y = Y\left(\frac{x}{D}, Fo, Bi\right)$$

1D Heat Transfer: Unsteady State

Unsteady State Heat Conduction in a Semi-Infinite Slab

$$\frac{\partial T}{\partial t} = \frac{k}{\rho C_p} \left(\frac{\partial^2 T}{\partial x^2} \right) = \alpha \left(\frac{\partial^2 T}{\partial x^2} \right)$$

Initial condition: $t = 0 \quad T = T_0 \quad \forall x$

Boundary conditions:

$$x = 0 \quad \frac{q_x}{A} = -k \frac{dT}{dx} = h(T_1 - T) \quad t > 0$$

$$x = \infty \quad T = T_0 \quad \forall t$$

thermal diffusivity $\alpha = \frac{k}{\rho C_p}$

Dimensionless quantities:

$$Y = \frac{(T_1 - T)}{(T_1 - T_0)}$$

Y (dimensionless temperature interval)

$$t^* = Fo = \frac{\alpha t}{D^2}$$

Fourier number (dimensionless time based on thermal diffusivity)

$$x^* = \frac{x}{D}$$

x* (dimensionless position)

$$Bi = \frac{hD}{k}$$

Biot number (pronounced BEE-OH)

Ratio of internal heat transfer resistance to resistance at the boundary. This is a *transport* issue.

13

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Because we can solve this problem analytically, we can confirm that the dimensional analysis is correct:

this:

Solution:

Unsteady State Heat Conduction in a Semi-Infinite Slab

Solution:

$$\frac{T - T_0}{T_1 - T_0} = \text{erfc} \zeta - e^{\beta(2\zeta + \beta)} \text{erfc}(\zeta + \beta)$$

$$\beta \equiv \frac{h\sqrt{\alpha t}}{k} \quad \zeta \equiv \frac{x}{2\sqrt{\alpha t}}$$

1D Heat Transfer: Unsteady State

Unsteady State Heat Conduction in a Semi-Infinite Slab

$$\frac{\partial T}{\partial t} = \frac{k}{\rho C_p} \left(\frac{\partial^2 T}{\partial x^2} \right) = \alpha \left(\frac{\partial^2 T}{\partial x^2} \right)$$

Initial condition: $t = 0 \quad T = T_0 \quad \forall x$

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$$x = \infty \quad T = T_0 \quad \forall t$$

thermal diffusivity $\alpha = \frac{k}{\rho C_p}$

$$Y \equiv \frac{(T_1 - T)}{(T_1 - T_0)}$$

$$1 - Y = \frac{(T - T_0)}{(T_1 - T_0)}$$

+ Bi – Biot Number = $\frac{hD}{k}$

Fo – Fourier Number = $\frac{\alpha t}{D^2}$

=

$$1 - Y = \text{erfc} \left(\frac{x}{D} \frac{1}{2\sqrt{Fo}} \right) - e^{\text{Bi} \left(\frac{x}{D} \right) + \text{Bi}^2 Fo} \text{erfc} \left(\sqrt{Fo} \left(\text{Bi} + \frac{x}{D} \frac{1}{Fo} \right) \right)$$

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Unsteady State Heat Transfer in a Body

Two Additional Dimensionless Numbers

Bi – Biot Number = $\frac{hD_{char}}{k}$

Quantifies the tradeoffs between the internal resistance D/k (due to conduction) and $1/h$, the resistance at the phase boundary (due to convection)

Fo – Fourier Number = $\frac{\alpha t}{D_{char}^2}$

Scales the time evolution of the temperature profile relative to the material's thermal properties, $\alpha = k/\rho\hat{C}_p$ (thermal diffusion time).

Dimensionless Numbers

<p><small>momentum energy mass</small></p> <p>Re – Reynolds = $\frac{\rho VD}{\mu} = \frac{VD}{\nu}$</p> <p>Fr – Froude = $\frac{V^2}{gD}$</p> <p>Pe – Péclet_N = RePr = $\frac{\hat{C}_p \rho VD}{k} = \frac{VD}{\alpha}$</p> <p>Pe – Péclet_m = ReSc = $\frac{VD}{D_{AB}}$</p>	<p>These numbers from the governing equations tell us about the relative importance of the terms they precede in the microscopic balances (scenario properties).</p>
<p>Pr – Prandtl = $\frac{\hat{C}_p \mu}{k} = \frac{\nu}{\alpha}$</p> <p>Sc – Schmidt = LePr = $\frac{\mu}{\rho D_{AB}} = \frac{\nu}{D_{AB}}$</p> <p>Le – Lewis = $\frac{\alpha}{D_{AB}}$</p>	<p>These numbers compare the magnitudes of the diffusive transport coefficients ν, α, D_{AB} (material properties).</p>
<p>f – Friction Factor = $\frac{F_{drag}}{(\frac{1}{2}\rho V^2)A_c}$</p> <p>Nu – Nusselt = $\frac{hD}{k}$</p> <p>Sh – Sherwood = $\frac{k_m D}{D_{AB}}$</p>	<p>These numbers are defined to help us build transport data correlations based on the fewest number of grouped (dimensionless) variables (scenario properties).</p>

15
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Dimensional Analysis in Unsteady State Heat Transfer

Note Two Different Numbers
with completely different purposes and meanings but **confusingly similar definitions**

Warning!

Bi – Biot Number = $\frac{hD}{k} = \frac{hD_{body}}{k_{body}}$

Quantifies the tradeoffs between the internal resistance to heat flow D/k (due to conduction) and the external resistance to heat flow at the boundary $1/h$ (due to convection) for a body in contact with a moving fluid.

Nu – Nusselt Number = $\frac{hD}{k} = \frac{hD_{flow}}{k_{fluid}}$

Dimensionless heat transfer coefficient in convection. Quantifies the physics in the moving fluid and how this results in a resistance to heat transfer, captured in the heat transfer coefficient.

16
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Dimensional Analysis in Heat Transfer

Note also:

$$t_{char,1} = \frac{D}{V}$$

(forced convection)

$$t_{char,2} = \frac{D^2}{\alpha} = \frac{D^2 \rho \hat{C}_p}{k}$$

(thermal diffusion)

$$\frac{\text{convective rate}}{\text{diffusive rate}} = \frac{1/t_{char,1}}{1/t_{char,2}} = \frac{V/D}{k/D^2 \rho \hat{C}_p} = \frac{\rho V D \hat{C}_p}{k} = Pe$$

Dimensionless Numbers

$Re - \text{Reynolds} = \frac{\rho V D}{\mu} = \frac{VD}{\nu}$ $Fr - \text{Froude} = \frac{V^2}{gD}$ $Pe - \text{Péclet}_h = RePr = \frac{\rho V D \hat{C}_p}{k} = \frac{VD}{\alpha}$ $Pe - \text{Péclet}_m = ReSc = \frac{VD}{D_{AB}}$	These numbers from the governing equations tell us about the relative importance of the terms they precede in the microscopic balances (scenario properties).
$Pr - \text{Prandtl} = \frac{\hat{C}_p \mu}{k} = \frac{\nu}{\alpha}$ $Sc - \text{Schmidt} = LePr = \frac{\mu}{\rho D_{AB}} = \frac{\nu}{D_{AB}}$ $Le - \text{Lewis} = \frac{\alpha}{D_{AB}}$	These numbers compare the magnitudes of the diffusive transport coefficients ν, α, D_{AB} (material properties).
$f - \text{Friction Factor} = \frac{F_{drag}}{(\frac{1}{2} \rho V^2) A_c}$ $Nu - \text{Nusselt} = \frac{hD}{k}$ $Sh - \text{Sherwood} = \frac{k_m D}{D_{AB}}$	These numbers are defined to help us build transport data correlations based on the fewest number of grouped (dimensionless) variables (scenario properties).

Pe – Péclet_h = RePr

Pr – Prandtl

The Peclet number is the ratio of convective and diffusive heat transport rates

Non-dimensional Energy Equation

$$\left(\frac{\partial T^*}{\partial t^*} + \mathbf{v}^* \cdot \nabla^* T^* \right) = \frac{1}{RePr} (\nabla^{*2} T^*) + S^*$$

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Bi – Biot Number = $\frac{hD}{k}$

$$Bi = \frac{\mathcal{R}_k}{\mathcal{R}_h} = \frac{D/k}{1/h}$$

Quantifies the tradeoffs between the resistance to heat flow (due to conductivity, D/k) and the resistance to heat flow at the boundary ($1/h$)

At high Bi, the surface temperature equals the bulk temperature; heat transfer is limited by conduction in the body.

At moderate Bi, heat transfer is affected by both conduction in the body and the rate of heat transfer to the surface.

At low Bi, the temperature is uniform in a finite body; heat transfer is limited by rate of heat transfer to the surface (h).

High Bi:
low k ,
high h

Moderate Bi:
neither process dominates

Low Bi:
high k ,
low h

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Bi – Biot Number $= \frac{hD}{k}$

Quantifies the tradeoffs between the resistance to heat flow (due to conductivity, D/k) and the resistance to heat flow at the boundary ($1/h$)

$Bi = \frac{D_{char}/k}{1/h}$

$D_{char} = \frac{V_{body}}{A_{body}}$

At high Bi, the surface temperature equals the bulk temperature; heat transfer is limited by rate of heat transfer to the surface.

At low Bi, the temperature is uniform in a finite body; heat transfer is limited by rate of heat transfer to the surface (h).

High Bi:
low k ,
high h

Low Bi:
high k ,
low h

dominates

When the temperature is uniform in the body, we can do a macroscopic energy balance to solve many problems of interest. This is called a “lumped parameter analysis.”

$T = T(t)$

19
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Bi – Biot Number $= \frac{hD}{k}$

Quantifies the tradeoffs between the resistance to heat flow (due to conductivity, D/k) and the resistance to heat flow at the boundary ($1/h$)

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At low Bi, the temperature is uniform in a finite body; heat transfer is limited by rate of heat transfer to the surface (h).

High Bi:
low k ,
high h

Moderate Bi:

Low Bi:
high k ,
low h

When the wall temperature and the bulk temperature are equal, the microscopic energy balance is easier to carry out (temperature boundary conditions).

$T = T(x, y, z, t)$,
easy BC

20
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Bi – Biot Number = $\frac{hD}{k}$

$Bi = \frac{D/k}{1/h}$

Quantifies the tradeoffs between the resistance to heat flow (due to conductivity, D/k) and the resistance to heat flow at the boundary ($1/h$)

When both processes affect the outcomes, the full solution may be necessary. For uniform starting temperatures, the solutions are published.

the body. $T = T(x, y, z, t)$

hard BC

At moderate Bi, heat transfer is affected by both conduction in the body and the rate of heat transfer to the surface.

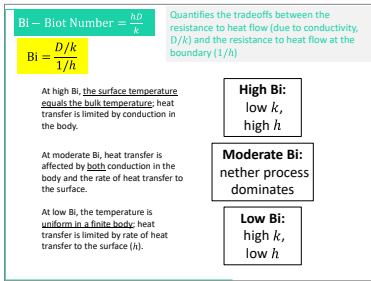
At low Bi, the temperature is uniform in a finite body; heat transfer is limited by rate of heat transfer to the surface (h).

Moderate Bi:
nether process dominates


Low Bi:
high k ,
low h

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NEXT: Lecture V (Talk about the three Bi cases)



But! Before dimensionless numbers get more out of hand: Library of Dimensionless Numbers



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11

Library of Dimensionless Numbers in Transport

Dimensionless numbers:

1. From the Equations of Change for
 - a. Momentum
 - b. Energy
 - c. Species A Mass
2. Comparing Transport Coefficients (material properties)
3. Involving Engineering Quantities of Interest (scenario properties)
4. Unsteady Heat Transfer

23

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Dimensional Analysis

These numbers tell us about the relative importance of the terms they precede.

Dimensionless numbers from the **Equations of Change** (microscopic balances)

momentum

Non-dimensional Navier-Stokes Equation

$$\left(\frac{\partial v_z^*}{\partial t^*} + \underline{v}^* \cdot \nabla^* v_z^* \right) = -\frac{\partial P^*}{\partial z^*} + \frac{1}{\text{Re}} (\nabla^{*2} v_z^*) + \frac{1}{\text{Fr}} g^*$$

Re – Reynolds
Fr – Froude

energy

Non-dimensional Energy Equation

$$\left(\frac{\partial T^*}{\partial t^*} + \underline{v}^* \cdot \nabla^* T^* \right) = \frac{1}{\text{RePr}} (\nabla^{*2} T^*) + S^*$$

Pe – Péclet_h = **RePr**
Pr – Prandtl

mass

Non-dimensional Continuity Equation (species A)

$$\left(\frac{\partial x_A^*}{\partial t^*} + \underline{v}^* \cdot \nabla^* x_A^* \right) = \frac{1}{\text{ReSc}} (\nabla^{*2} x_A^*)$$

Pe – Péclet_m = **ReSc**
Sc – Schmidt

ref: BSL1, p581, 644

24

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Dimensionless Numbers

Dimensionless numbers from the **Equations of Change**

Re – Reynolds = $\frac{\rho V D}{\mu} = \frac{V D}{\nu}$

Fr – Froude = $\frac{v^2}{g D}$

Pe – Péclet_h = **RePr** = $\frac{\hat{c}_p \rho V D}{k} = \frac{V D}{\alpha}$

Pe – Péclet_m = **ReSc** = $\frac{V D}{D_{AB}}$

Pr – Prandtl = $\frac{\hat{c}_p \mu}{k} = \frac{\nu}{\alpha}$

Sc – Schmidt = **LePr** = $\frac{\mu}{\rho D_{AB}} = \frac{\nu}{D_{AB}}$

Le – Lewis = $\frac{\alpha}{D_{AB}}$

These numbers tell us about the **relative importance of the terms** they precede in the microscopic balances (*scenario properties*).

These numbers compare the **magnitudes of the diffusive transport coefficients** ν, α, D_{AB} (*material properties*).

25
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Dimensionless Numbers

Dimensionless numbers from the **Equations of Change**

Re – Reynolds = $\frac{\rho V D}{\mu} = \frac{V D}{\nu}$

Fr – Froude = $\frac{v^2}{g D}$

Pe – Péclet_h = **RePr** = $\frac{\hat{c}_p \rho V D}{k} = \frac{V D}{\alpha}$

Pe – Péclet_m = **ReSc** = $\frac{V D}{D_{AB}}$

Pr – Prandtl = $\frac{\hat{c}_p \mu}{k} = \frac{\nu}{\alpha}$

Sc – Schmidt = **LePr** = $\frac{\mu}{\rho D_{AB}} = \frac{\nu}{D_{AB}}$

Le – Lewis = $\frac{\alpha}{D_{AB}}$

These numbers tell us about the **relative importance of the terms** they precede in the microscopic balances (*scenario properties*).

These numbers compare the **magnitudes of the diffusive transport coefficients** ν, α, D_{AB} (*material properties*).

Transport coefficients

26
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Dimensional Analysis

$Nu_{AB} =$

These numbers are defined to help us build **transport data correlations** based on the fewest number of grouped (dimensionless) variables (*scenario property*).

Dimensionless numbers from the Engineering Quantities of Interest

momentum	<p style="font-size: small; margin: 0;">Dimensionless Force on the Wall (Drag)</p> $f = \frac{1}{2} \frac{D}{L} \frac{1}{Re} \int_0^{\frac{D}{2}} \int_0^{2\pi} \left(\frac{\partial v_z}{\partial r^*} \right) \Big _{r^*=\frac{1}{2}} d\theta dz^*$	<p style="font-size: small; margin: 0;">(Fanning)</p> f – Friction Factor $\frac{L}{D}$ – Aspect Ratio	$f = \frac{\mathcal{F}_{drag}}{\left(\frac{1}{2} \rho V^2\right) A_c}$
energy	<p style="font-size: small; margin: 0;">Newton's Law of Cooling</p> $Nu = \frac{1}{2\pi L / D} \int_0^{\frac{D}{2}} \int_0^{2\pi} \left(\frac{\partial T^*}{\partial r^*} \right) \Big _{r^*=\frac{1}{2}} dz^* d\theta$	Nu – Nusselt $\frac{L}{D}$ – Aspect Ratio	$Nu = \frac{hD}{k}$
mass xfer	<p style="font-size: small; margin: 0;">Dimensionless Mass Transfer Coefficient</p> $Sh = \frac{1}{2\pi L} \int_0^{\frac{D}{2}} \int_0^{2\pi} \left(-\frac{\partial x_A}{\partial r^*} \right) \Big _{r^*=\frac{1}{2}} d\theta dz^*$	$Nu_{AB} = Sh$ – Sherwood $\frac{L}{D}$ – Aspect Ratio	$Sh = \frac{k_m D}{D_{AB}} = Nu_{AB}$

27

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momentum
 energy
 mass species A

Dimensionless Numbers

Re – Reynolds = $\frac{\rho V D}{\mu} = \frac{V D}{\nu}$ Fr – Froude = $\frac{V^2}{g D}$ Pe – Péclet _h = $RePr = \frac{\hat{c}_p \rho V D}{k} = \frac{V D}{\alpha}$ Pe – Péclet _m = $ReSc = \frac{V D}{D_{AB}}$	<p>These numbers from the governing equations tell us about the relative importance of the terms they precede in the microscopic balances (<i>scenario properties</i>).</p>
Pr – Prandtl = $\frac{\hat{c}_p \mu}{k} = \frac{\nu}{\alpha}$ Sc – Schmidt = $LePr = \frac{\mu}{\rho D_{AB}} = \frac{\nu}{D_{AB}}$ Le – Lewis = $\frac{\alpha}{D_{AB}}$	<p>These numbers compare the magnitudes of the diffusive transport coefficients ν, α, D_{AB} (<i>material properties</i>).</p>
f – Friction Factor = $\frac{\mathcal{F}_{drag}}{\left(\frac{1}{2} \rho V^2\right) A_c}$ Nu – Nusselt = $\frac{hD}{k}$ Sh – Sherwood = $\frac{k_m D}{D_{AB}} = Nu_{AB}$	<p>These numbers are defined to help us build transport data correlations based on the fewest number of grouped (dimensionless) variables (<i>scenario properties</i>).</p>

28

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Dimensionless Numbers (Unsteady Heat Transfer)

$$\text{Bi} - \text{Biot} = \frac{hD_{char}}{k}$$

Ratio of internal resistance to external resistance to heat transfer

$$\text{Fo} - \text{Fourier} = \frac{\alpha D}{V^2}$$

Dimensionless time

$$\text{Pe} - \text{Péclet}_h = \frac{\hat{c}_p \rho V D}{k} = \frac{VD}{\alpha} = \frac{t_{convection}}{t_{thermal\ diffusion}}$$

Ratio of convective timescale to thermal diffusion time scale