

CM3215

MichiganTech

Fundamentals of Chemical Engineering Laboratory

Pumping System Curves

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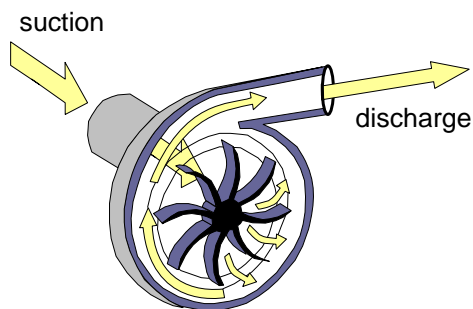
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Centrifugal Pumps

- Centrifugal force is used to fling fluid from the suction side to the discharge
- Centrifugal pumps put out **neither** constant flow rate **nor** constant pressure
- We must use the mechanical energy balance to figure out how a centrifugal pump will perform in a given situation



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System Curve Assignment (week 9; due Wed after break)

How do you choose a centrifugal pump for a given duty?

- Calculate the flow-rate-dependent **demands** of a system = **system head curve** (this assignment)
- Compare the system-head curve (demands) to the available pumping-head curve (**supply**), and choose the right pump

Pumping Head Lab (week 10)

- Pumping Head Characteristic Curves are plots of what an existing pump can do under various loads (duties)
- We measure the pump characteristic curve by determining $\Delta p = p_{discharge} - p_{suction}$ on the suction/discharge system

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Discuss now

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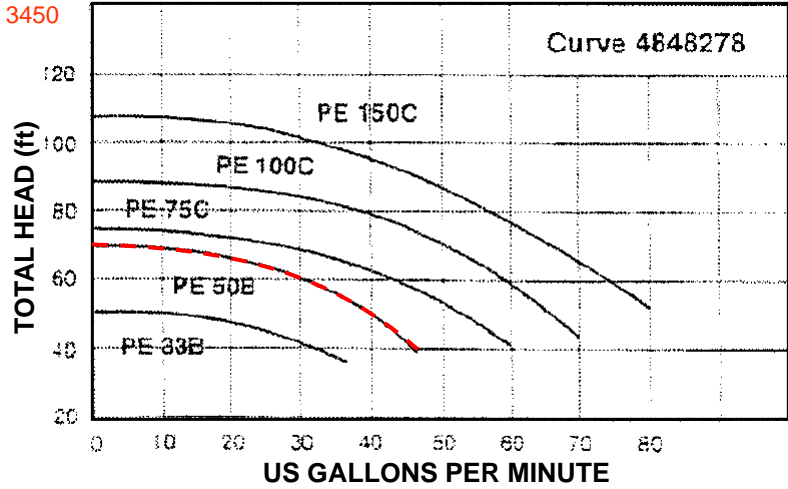
Pumping Head Lab (week 10)

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Krum Pump Company
 Kalamazoo, MI
 Model: Peerless pump
 Type: PE50B
 Performance Curve No: 4848278
 RPM: 3450



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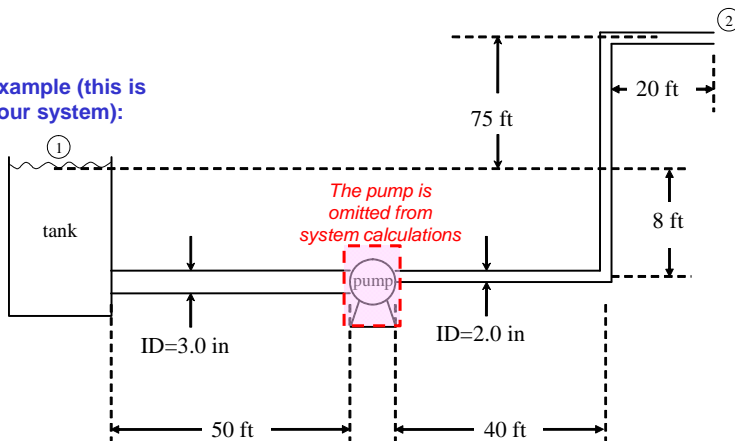
System Head - MEB written on total system, excluding pump

DEMAND

$$\underbrace{\frac{\Delta P}{\rho g} + \frac{\Delta(v^2)}{2g\alpha} + \Delta z + \frac{F_{friction}}{g}}_{\text{DEMAND}} = \underbrace{\frac{W_{s,on}}{\dot{m}g}}_{\text{SUPPLY}}$$

$$\frac{\Delta P}{\rho g} + \frac{\Delta(v^2)}{2g\alpha} + \Delta z + \frac{F_{friction}}{g} = H_{system}$$

For example (this is NOT our system):



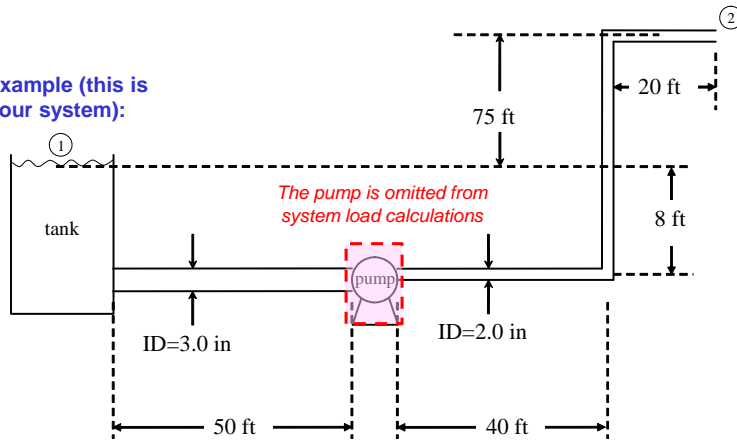
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System Head - MEB written on total system, excluding pump

The pump must overcome pressure, velocity, elevation, and friction **both** when it pulls upstream fluid towards it **and** while it pushes downstream fluid away from it

For example (this is NOT our system):



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System Head - MEB written on total system, excluding pump

DEMAND

$$H_{system} \equiv \frac{\Delta p}{\rho g} + \frac{\Delta \langle v \rangle^2}{2g\alpha} + \Delta z + \frac{F_{friction}}{g}$$


$$H_{system} = \frac{p_{finish} - p_{start}}{\rho g} + \frac{\langle v \rangle_{finish}^2 - \langle v \rangle_{start}^2}{2g\alpha} + (z_{finish} - z_{start}) + \frac{F_{finish,start}}{g}$$

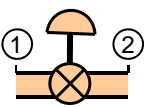
For a system that is not yet built, how can we estimate these frictional loads on the pump?


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MEB on Fittings and Straight Pipe

Straight pipe:  $\frac{\Delta p}{\rho g} + \frac{\Delta \langle v \rangle^2}{2g\alpha} + \Delta z + \frac{F_{friction}}{g} = \frac{W_{s,on}}{\dot{m}g}$

valve:  $\frac{\Delta p}{\rho g} + \frac{\Delta \langle v \rangle^2}{2g\alpha} + \Delta z + \frac{F_{friction}}{g} = \frac{W_{s,on}}{\dot{m}g}$

90° bend:  $\frac{\Delta p}{\rho g} + \frac{\Delta \langle v \rangle^2}{2g\alpha} + \Delta z + \frac{F_{friction}}{g} = \frac{W_{s,on}}{\dot{m}g}$

etc.

$$\frac{F_{finish,start}}{g} = \sum_{\substack{\text{fittings,} \\ \text{straight pipe, } i}} \left(\frac{\Delta p}{\rho g} \right)_i$$

Friction manifests as $\Delta p / \rho g$ for each fitting or pipe, which can be added up

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System Head - MEB written on total system, excluding pump

DEMAND

$$H_{system} \equiv \frac{\Delta p}{\rho g} + \frac{\Delta \langle v \rangle^2}{2g\alpha} + \Delta z + \frac{F_{friction}}{g}$$

$$H_{system} = \frac{p_{finish} - p_{start}}{\rho g} + \frac{\langle v \rangle_{finish}^2 - \langle v \rangle_{start}^2}{2g\alpha} + (z_{finish} - z_{start}) + \frac{F_{finish,start}}{g}$$

$$\frac{F_{finish,start}}{g} = \sum_{\substack{\text{fittings,} \\ \text{straight pipe}}} \left(\frac{\Delta p}{\rho g} \right)_i$$

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Friction in Fittings, Straight Pipe: Data Correlations from Literature

$$\frac{F_{finish,start}}{g} = \sum_{\substack{\text{fittings,} \\ \text{straight pipe}}} \left(\frac{\Delta p}{\rho g}\right)_i$$

These have been measured and correlated in the literature as a function of flow rate through Fanning friction factor $f(Re)$ (straight pipes) and K_f (fittings)

$$\frac{F_{finish,start}}{g} = \sum_{\substack{\text{fittings,} \\ \text{straight pipe}}} \left(\frac{\Delta p}{\rho g}\right)_i \quad \langle v \rangle = \frac{Q}{\pi R^2}$$

$$= \left(4f \frac{L}{D} + \sum_{i \text{ fittings}} K_{f_i} n_i \right) \frac{\langle v \rangle^2}{2g}$$

Note: if diameter changes, $\langle v \rangle$ changes; thus we need separate calculations for every $\langle v \rangle$

Data correlation for friction factor Δp versus Re (flow rate) in a pipe

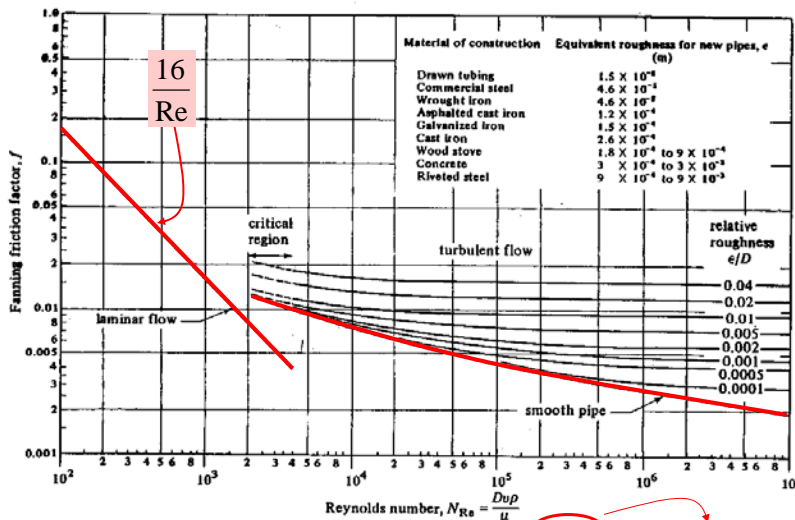
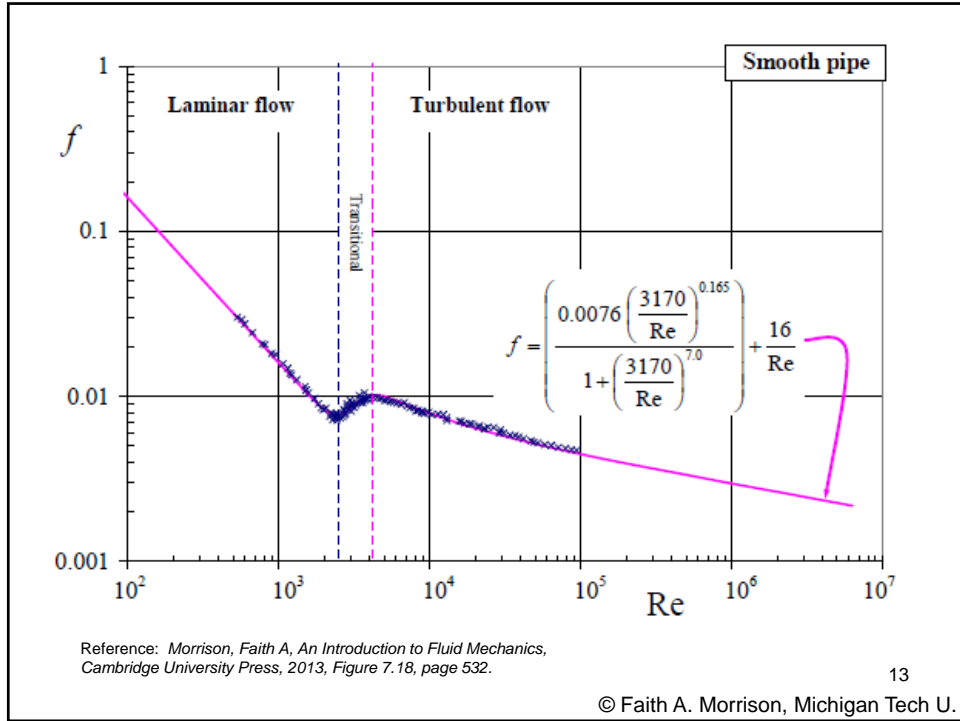


FIGURE 2.10-3. Friction factors for fluids inside pipes. [Based on L. F. Moody, trans. A.S.M.E., 66, 671, (1944); Mech. Eng., 1005 (1949). With permission.]

Moody Chart

(Geankoplis)



DEMAND

$$H_{system} = \frac{p_{finish} - p_{start}}{\rho g} + \frac{\langle v \rangle_{finish}^2 - \langle v \rangle_{start}^2}{2g\alpha} + (z_{finish} - z_{start}) + \frac{F_{finish,start}}{g}$$

Some terms go as Q^2

$$F_{finish,start} = \left(4f \frac{L}{D} + \sum_{i \text{ fittings}} K_{f_i} n_i \right) \frac{\langle v \rangle^2}{2}$$

(slight additional Q dependence)

$$\langle v \rangle = \frac{Q}{\pi R^2}$$

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DEMAND

$$H_{system} = \frac{p_{finish} - p_{start}}{\rho g} + \frac{\langle v \rangle_{finish}^2 - \langle v \rangle_{start}^2}{2g\alpha} + (z_{finish} - z_{start}) + \frac{F_{finish,start}}{g}$$

Some terms go as Q^0

$$F_{finish,start} = \left(4f \frac{L}{D} + \sum_{i \text{ fittings}} K_{f_i} n_i \right) \frac{\langle v \rangle^2}{2}$$

$$\langle v \rangle = \frac{Q}{\pi R^2}$$

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DEMAND

$$H_{system} = \frac{p_{finish} - p_{start}}{\rho g} + \frac{\langle v \rangle_{finish}^2 - \langle v \rangle_{start}^2}{2g\alpha} + (z_{finish} - z_{start}) + \frac{F_{finish,start}}{g}$$

$$F_{finish,start} = \left(4f \frac{L}{D} + \sum_{i \text{ fittings}} K_{f_i} n_i \right) \frac{\langle v \rangle^2}{2}$$

$H_{system} = aQ^2 + b$

$H_{system}(Q)$ is the system head curve

$$\langle v \rangle = \frac{Q}{\pi R^2}$$

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DEMAND

System Head - MEB written on total system, excluding pump

$H_{system}(Q)$ is the system head curve

$$H_{system} = aQ^2 + b$$

Q (gal/min)

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DEMAND

$$H_{system} = \frac{p_{finish} - p_{start}}{\rho g} + \frac{\langle v \rangle_{finish}^2 - \langle v \rangle_{start}^2}{2g\alpha} + (z_{finish} - z_{start}) + \frac{F_{finish,start}}{g}$$

Calculating the System curve:

- Choose two points that enclose the entire system from start to finish
- Write pressures, elevations, velocities at finish and start
- Write velocities in terms of flow rate Q
- Calculate the friction of all piping, fittings, devices between start and finish as a function of Q
- **Do not include a pump** (we are calculating what expected load the as-yet-unchosen pump must overcome)

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Assignment 5: Calculate and plot the system head curves for the assigned system.

(come into lab to get measurements you need)

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CM3215 Assignment 5: System-head Curves for a Piping System Under Conditions of Different Valve Positions

Complete all calculations described below; you may verbally consult with any of your classmates, but you must submit individual assignments. Deliver your submission with a memo of transmittal that clearly lists where to find your submitted answers to the five assigned objectives.

Overall objective: Determine the equations for the system-head curves for three different systems. Plot these curves.

1. Sketch the following system using Visio or by hand (make the sketch the size of an entire sheet of paper):

The system: All the pipes, fittings, and devices between the exit of the orifice meter (point A) and the pressure tap on the suction side of the Krum centrifugal pump (point B). Include the sizes and locations of all valves, pressure taps, thermo wells,

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What's this good for?

(choosing pumps)

How do we use the system head curve to choose a pump?

Plot this versus Q

$$H_{system} = \frac{\Delta P}{\rho g} + \frac{\Delta(v^2)}{2g\alpha} + \Delta z + \frac{F_{friction}}{g} = H_{sys}$$

All the contributions to load on the pump = H_{sys} system head

DEMAND

$$\frac{W_{s,on}}{\dot{m}g} \equiv H_{pump}$$

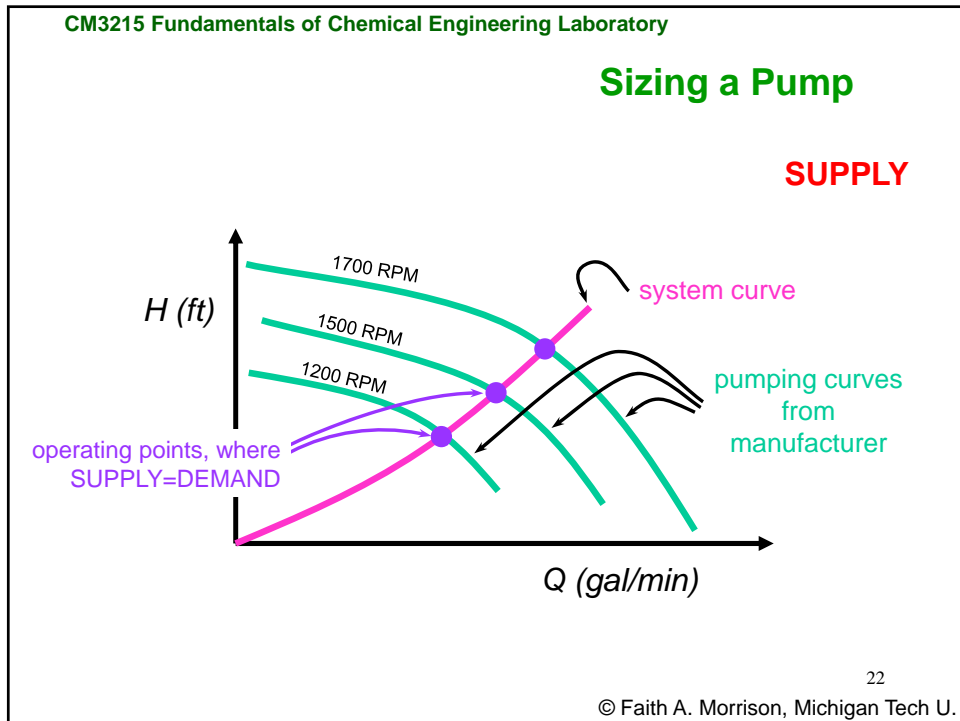
Pumping effort and friction are accounted for when the system curve intersects the pumping head curve $H_{pump}(Q)$

SUPPLY

Obtain this versus Q from the manufacturer

Where the two intersect, that's where the pump operates.

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WEEK 10

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The CM3215 “Lossy” Pump

- Extends from the pump suction to the exit of the orifice meter.
- The system on which this pump operates thus is the piping and fittings between the exit of the orifice meter and the pump suction

