

12.6 - Defining the Transient Stability Problem

- Assumptions:
- Ignore magnetic saturation
 - Assume machine is lossless.
 - Assume balanced 3 ϕ operation.
 - Assume P_m constant
 - Assume $|E_f|$ constant

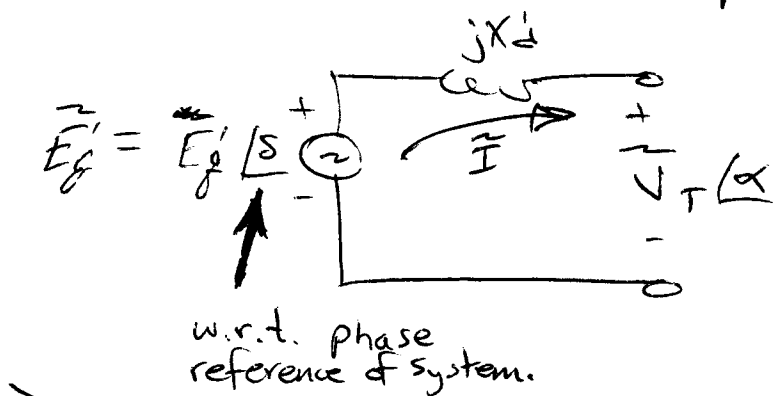
For salient pole machine:

$$E_f = \bar{V}_T + jX_f \bar{I}_f + jX'_d \bar{I}_d = \bar{E}_f + j(X_d - X_f) \bar{I}_d$$

Recall $\bar{I} = \bar{I}_d + \bar{I}_q$ (see fig. p. 515 in Gross)

For stability calculations, use X'_d .

Illustration on p. 515 shows that if $X'_d \approx X_f$ then machine can be simplified to



$$P_e = \frac{E'_f V}{X'_d} \sin \delta \quad | \quad X'_d = X_f$$

If $X'_d \neq X_f$,

$$P_e = \frac{E'_f V}{X'_d} \sin \delta + \frac{V^2}{2} \left(\frac{1}{X_f} + \frac{1}{X'_d} \right) \sin 2\delta$$

at $t = 0^-$, prior to disturbance.

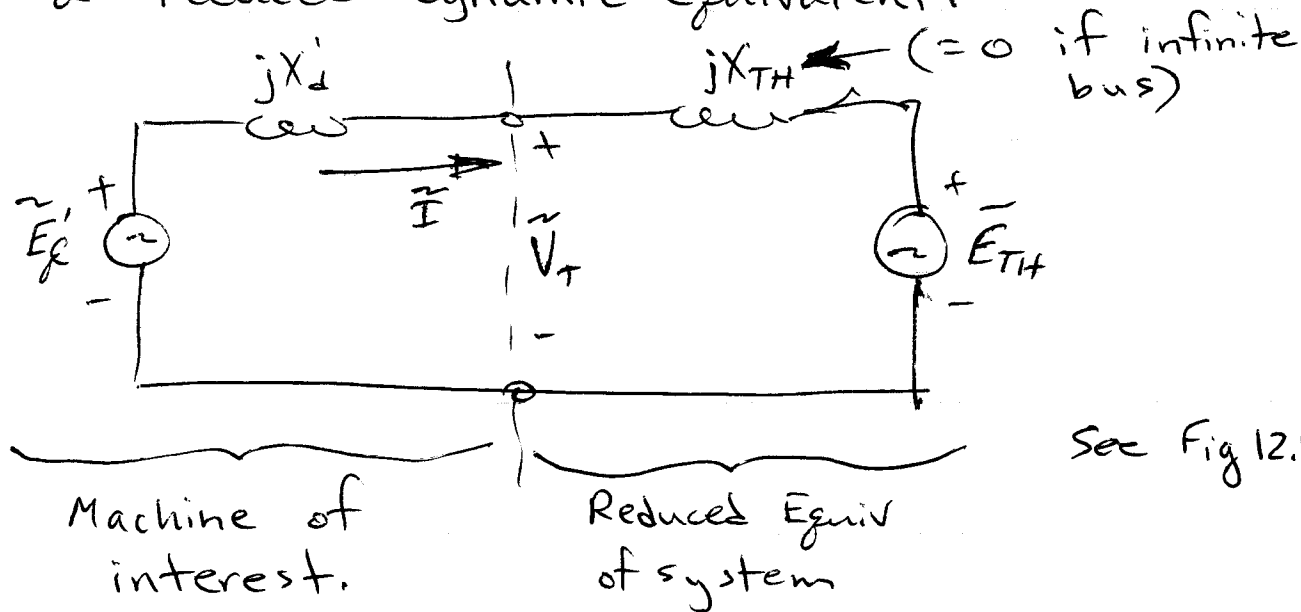
EE406

Infinite Bus - Ideal voltage source of constant frequency

In terms of a generator, this means:

- ⇒
- Constant Internal voltage
 - Zero impedance
 - Infinite Inertia ($\omega_{ref} = \omega_s$)

When considering how a disturbance affects one particular machine, it is possible to approximate the remainder of the system as a reduced dynamic equivalent!



If response does not involve governor or exciter, the relationship

$$P_m - P_e = \frac{H}{\pi f} \left(\frac{d^2\delta}{dt^2} + \frac{d\omega_{ref}}{dt} \right)$$

if infinite bus

can be used. This assumes that $\omega_{ref} = \omega_s$.

To evaluate,

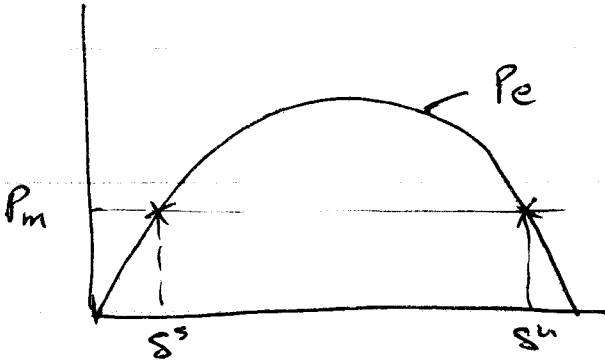
$$P_m = \text{constant}$$

$$P_e = \frac{E'_g E_{TH}}{X'_d + X_{TH}} \sin \delta$$

Equilibria Points:

when $P_e = P_m$

$$P_m - P_e = \frac{H}{\pi f} \frac{d^2 \delta}{dt^2}$$



Perturbations:

δ^s is stable

From δ^s , if δ increases, $\frac{d^2 \delta}{dt^2} < 0$ ($P_m < P_e$)
 $\Rightarrow \delta$ decreases
 if δ decreases, $\alpha > 0$ ($P_m > P_e$)
 $\Rightarrow \delta$ increases.

δ^u is unstable

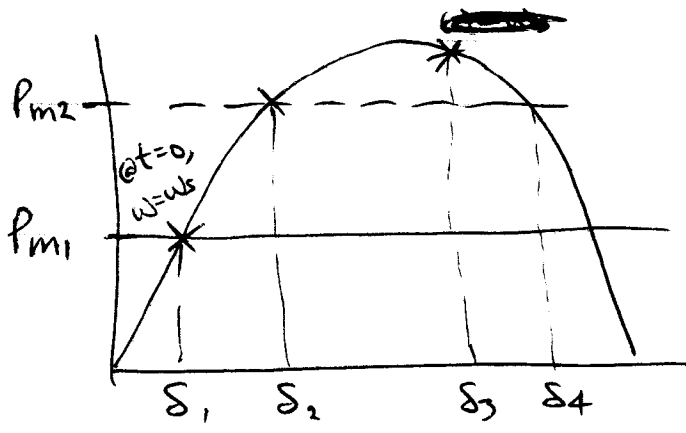
From δ^u , if δ increases, $\alpha > 0$ ($P_m > P_e$)
 $\Rightarrow \delta$ increases further.
 if δ decreases, $\alpha < 0$ ($P_m < P_e$)

P_e proportional to $\sin \delta$ ↑ acceleration

\therefore Better keep $\delta < 90^\circ$ for stable equilibrium.

Conceptual example:

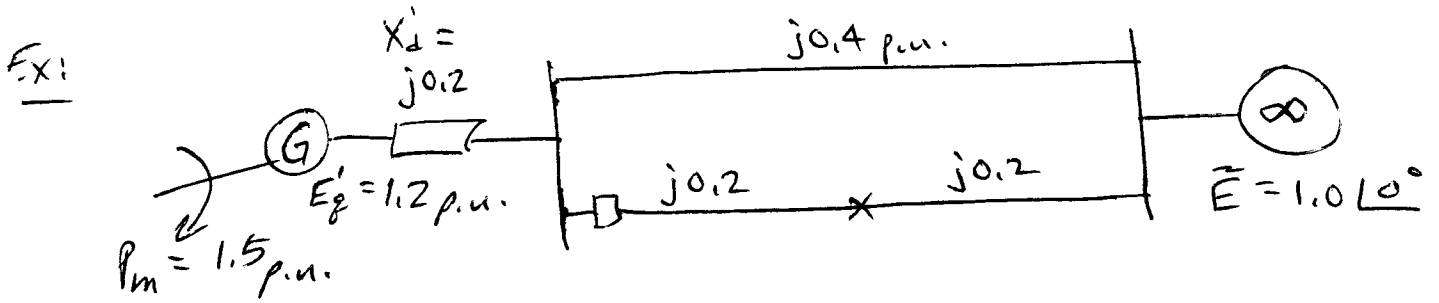
If P_m is initially P_{m1} , then suddenly increases to P_{m2} . What happens?



For $t \geq 0^+$, rotor accelerates, until δ reaches δ_2 , where it begins to decelerate. Oscillation in δ (about δ_2) continues, damping slowly.

$\delta < \delta_2 \Rightarrow P_m > P_e \Rightarrow$ accel
 $\delta > \delta_2 \Rightarrow P_m < P_e \Rightarrow$ decel
 $\delta > \delta_4 \Rightarrow P_m > P_e \Rightarrow$ accel

No chance to maintain stability if δ goes past δ_4 .

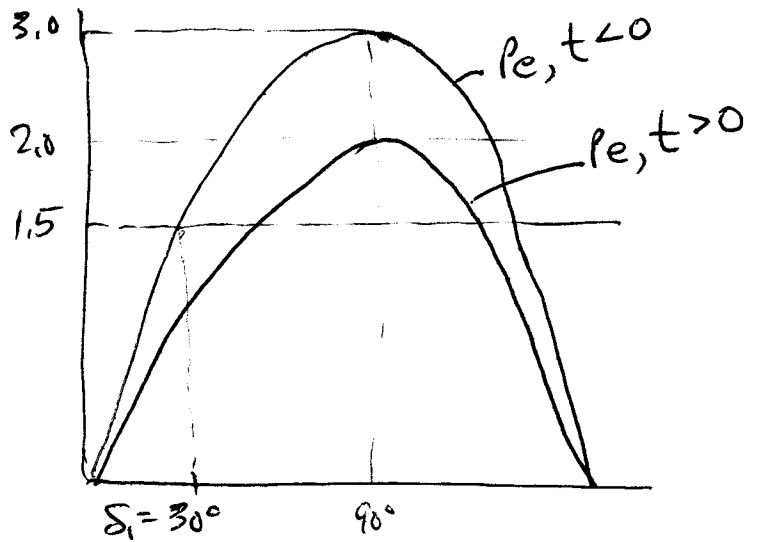


@ $t=0$ $X = j0.2$ p.u. (X refers to T-Line tie)

$$P_m = P_e = \frac{(1.2)(1.0)}{0.2+0.2} \sin \delta = 1.5 \text{ p.u.} \Rightarrow \delta_1 = 30^\circ$$

$= 3.0 = P_{e \max}$

~~Power~~



Breaker opens at $t=0$, ~~breaker~~

$$P_{\max} = \frac{(1.2)(1.0)}{0.2+0.4} = 2.0 \text{ p.u.}$$

New equilibrium point is at

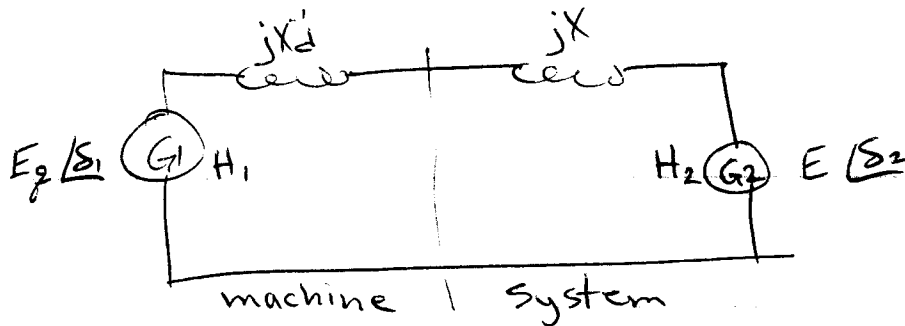
$$1.5 = P_{\max} \sin \delta = 2.0 \sin \delta$$

$$\sin \delta = 0.75 \Rightarrow \underline{\underline{\delta = 48.6^\circ}} \text{ or } \cancel{131.4^\circ} \text{ unstable}$$

Considerations / Questions

1) What if system equivalent cannot be assumed to be infinite bus (i.e. infinite inertia)?

This is a case where machines are noncoherent.



See pp. 705-706 in Grainger/Stevenson ©1994

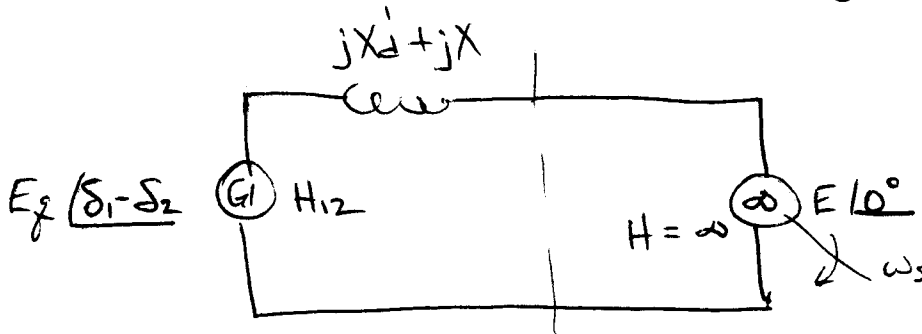
$$H_{12} = \frac{H_1 H_2}{H_1 + H_2} \Rightarrow$$

$$P_m - P_e = \frac{H_{12}}{\pi f} \frac{d^2 \delta_{12}}{dt^2}$$

where $P_m = P_{m1} = -P_{m2}$

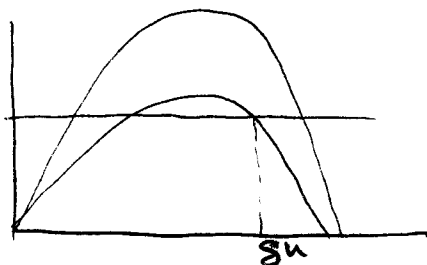
$P_e = P_{e1} = -P_{e2}$

$\delta_{12} = \delta_1 - \delta_2$



\therefore Convert system equiv to ∞ bus.

2) How do we know if δ swings past δ_u in new P_{max} curve? All we know so far is how to find new s-s equilibrium point.



Focus on this for remainder of Ch. 12.

12.7 - EQUAL AREA CRITERION

Back to swing equation:
$$P_m - P_e = \frac{H}{\pi f} \frac{d^2 \delta}{dt^2}$$

Rearranging,
$$\frac{d^2 \delta}{dt^2} = \frac{\pi f}{H} (P_m - P_e) = \frac{\pi f}{H} \left[P_m - \frac{E' V}{X} \sin \delta \right]$$

Because of $\sin \delta$ term in P_e , this is nonlinear differential equation. How to solve?

One "trick" that works:
$$\frac{d}{dt} \left(\frac{d\delta}{dt} \right)^2 = 2 \frac{d\delta}{dt} \left(\frac{d^2 \delta}{dt^2} \right)$$

We can substitute:
$$\frac{d^2 \delta}{dt^2} = \frac{1}{2} \frac{d}{dt} \left(\frac{d\delta}{dt} \right)^2 \frac{dt}{d\delta}$$

$$\frac{1}{2} \frac{d}{dt} \left(\frac{d\delta}{dt} \right)^2 \frac{dt}{d\delta} = \frac{\pi f}{H} (P_m - P_e)$$

$$\frac{d}{dt} \left(\frac{d\delta}{dt} \right)^2 = \frac{2\pi f}{H} (P_m - P_e) \frac{d\delta}{dt}$$

Integrating both sides ~~with respect to t~~,

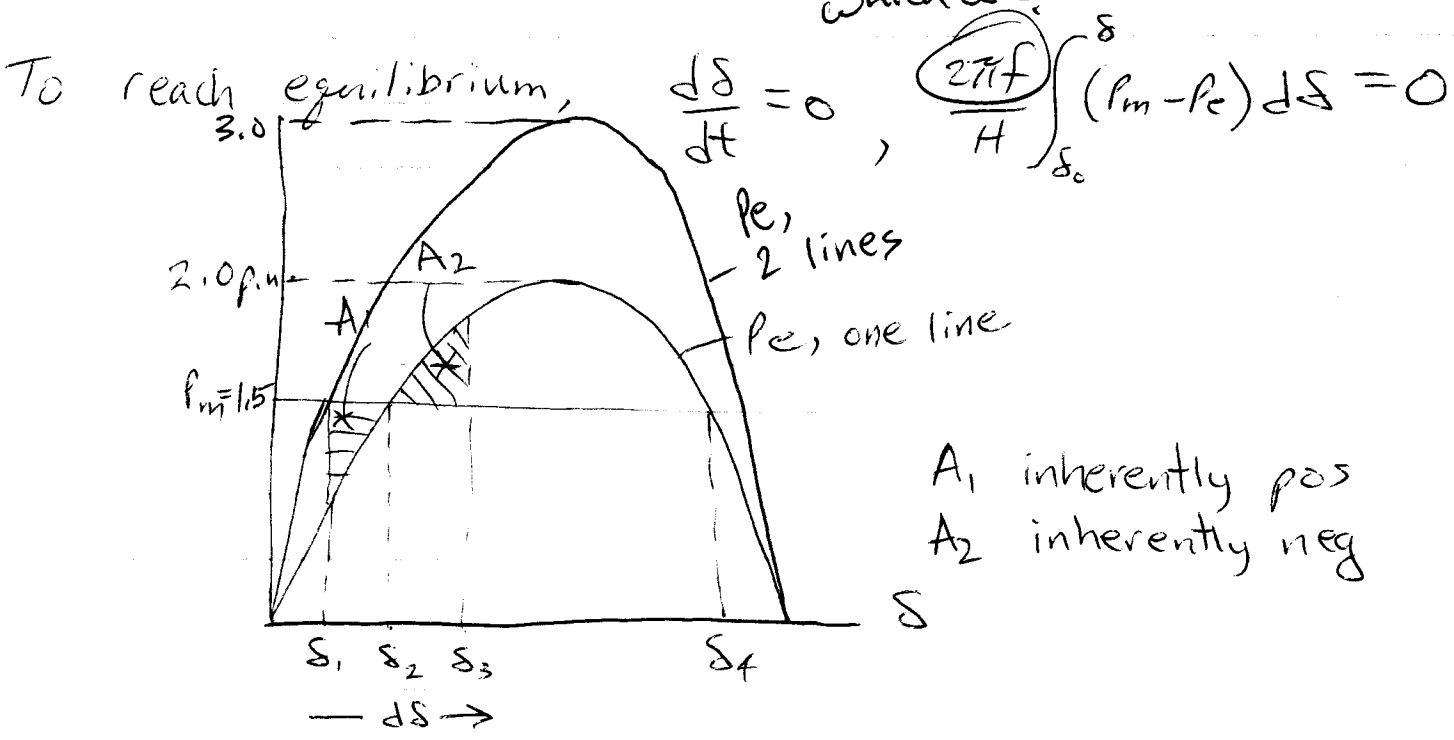
$$\left(\frac{d\delta}{dt} \right)^2 = \frac{2\pi f}{H} \int_{\delta_0}^{\delta} (P_m - P_e) d\delta = (\omega - \omega_s)^2$$

($\omega_{ref} = \omega_s$)

\Rightarrow Area between P_m & P_e ($\Delta P \cdot d\delta$) is proportional to relative velocity squared, $(\omega - \omega_{ref})^2$

Recall that kinetic energy $K.E. \propto \omega^2$ also, so areas are proportional to changes in kinetic energy. If $P_m > P_e$ then area is positive and energy increases (ω increases). If $P_m < P_e$ then ω decreases and energy decreases.

\therefore When $P_m > P_e$, energy stored in rotor ($\frac{1}{2}J\omega^2$),
 When $P_e < P_m$, energy released from rotor.
 Energy fluctuations are taken up by infinite bus.



Example: $\delta_0 = \delta_1 = 30^\circ$ (release from δ_1 at $t=0$).
 Clearly, equilibrium point will be $\delta = \delta_2$.
 For $P_{e\max} = 2.0$ & $P_m = 1.5$, $\delta_2 = 48.6^\circ = 0.848 \text{ rad}$

For equilibrium/stability,

$$\frac{2\pi f}{H} \int_{\delta_1}^{\delta_3} (P_m - P_e) d\delta = 0 \Rightarrow \underbrace{\int_{\delta_1}^{\delta_2} (P_m - P_e) d\delta}_{A_1} = \underbrace{\int_{\delta_2}^{\delta_3} (P_m - P_e) d\delta}_{A_2}$$

δ_2 pos pos δ_2 neg neg

$$A_1 = \int_{0.524}^{0.848} (1.5 - 2 \sin \delta) d\delta = 1.5\delta + 2 \cos \delta \Big|_{0.524}^{0.848} = \underline{\underline{0.0773}}$$

Since equilibrium is (hopefully) about δ_2 , A_2 will be equal to A_1 .

$$\therefore \int_{\delta_3}^{\delta_2} (P_m - P_e) d\delta = \int_{\delta_3}^{0.848} (1.5 - 2 \sin \delta) d\delta = 0.0773$$

$$1.5\delta + 2 \cos \delta \Big|_{\delta_3}^{0.848} = 0.0773$$

$$(1.5)(0.848) + 2 \cos(0.848) - 1.5(\delta_3) - 2 \cos(\delta_3) = 0.0773$$

$$2 \cos \delta_3 + 1.5\delta_3 = 2.518$$

solve? use Newton Raphson, iteration, etc.

$$\delta_3 = 1.218 \text{ rad } (69.8^\circ)$$

Note: Max A_2 available to "counter" A_1 is

$$A_2 = \int_{\delta_4}^{\delta_2} (P_m - P_e) d\delta = \int_{2.293}^{0.848} (1.5 - 2 \sin \delta) d\delta = \underline{\underline{0.478}}$$

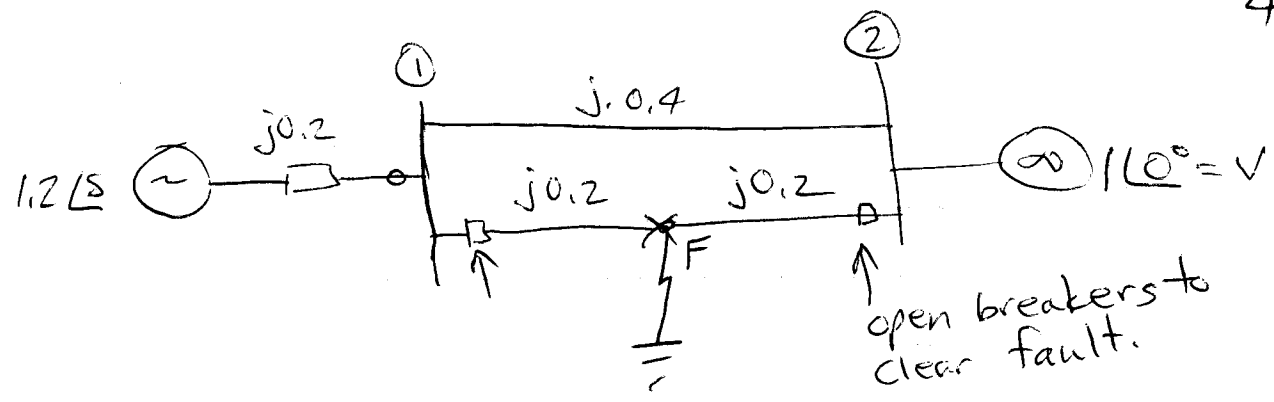
$$\text{IF } \delta_1 \text{ had been } 0^\circ, \quad A_1 = \int_0^{0.524} (1.5 - 2 \sin \delta) d\delta$$

$$= 1.5(0.524) + 2 \cos 0.524 - 2$$

$$= \underline{\underline{0.5176}}$$

\therefore Stability would not have been possible!

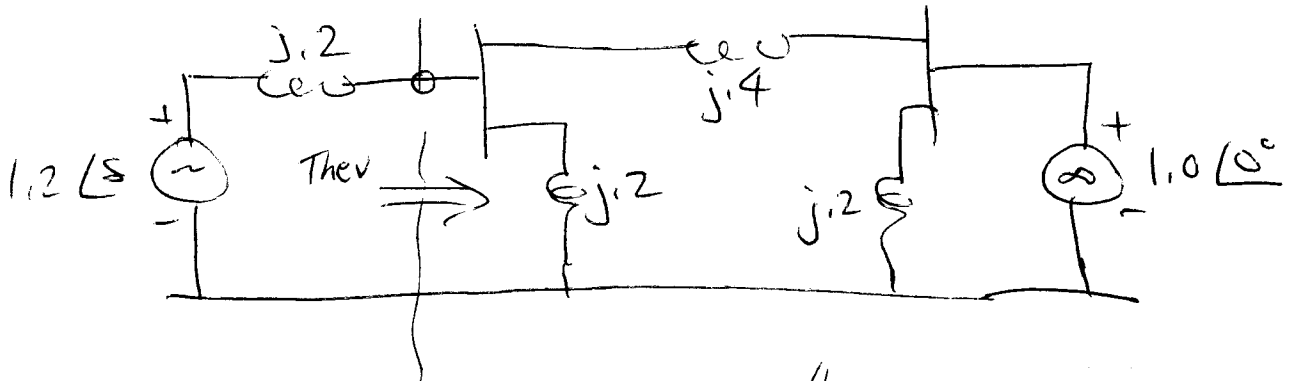
Ex:



What if fault occurs at $t=0$?

$$P_{e_{max}}(t=0^-) = \frac{(1.2)(1)}{j0.2 + j0.4 // j0.4} = \frac{1.2}{0.4} = 3.0 \text{ p.u.}$$

For fault, Thevenize at bus 1 look back to inf. bus.



$$Z_{TH} = j0.4 // j0.2 = \cancel{j0.1333} j0.1333$$

$$V_{TH} = 1 \left(\frac{j0.2}{j0.6} \right) = 0.333$$

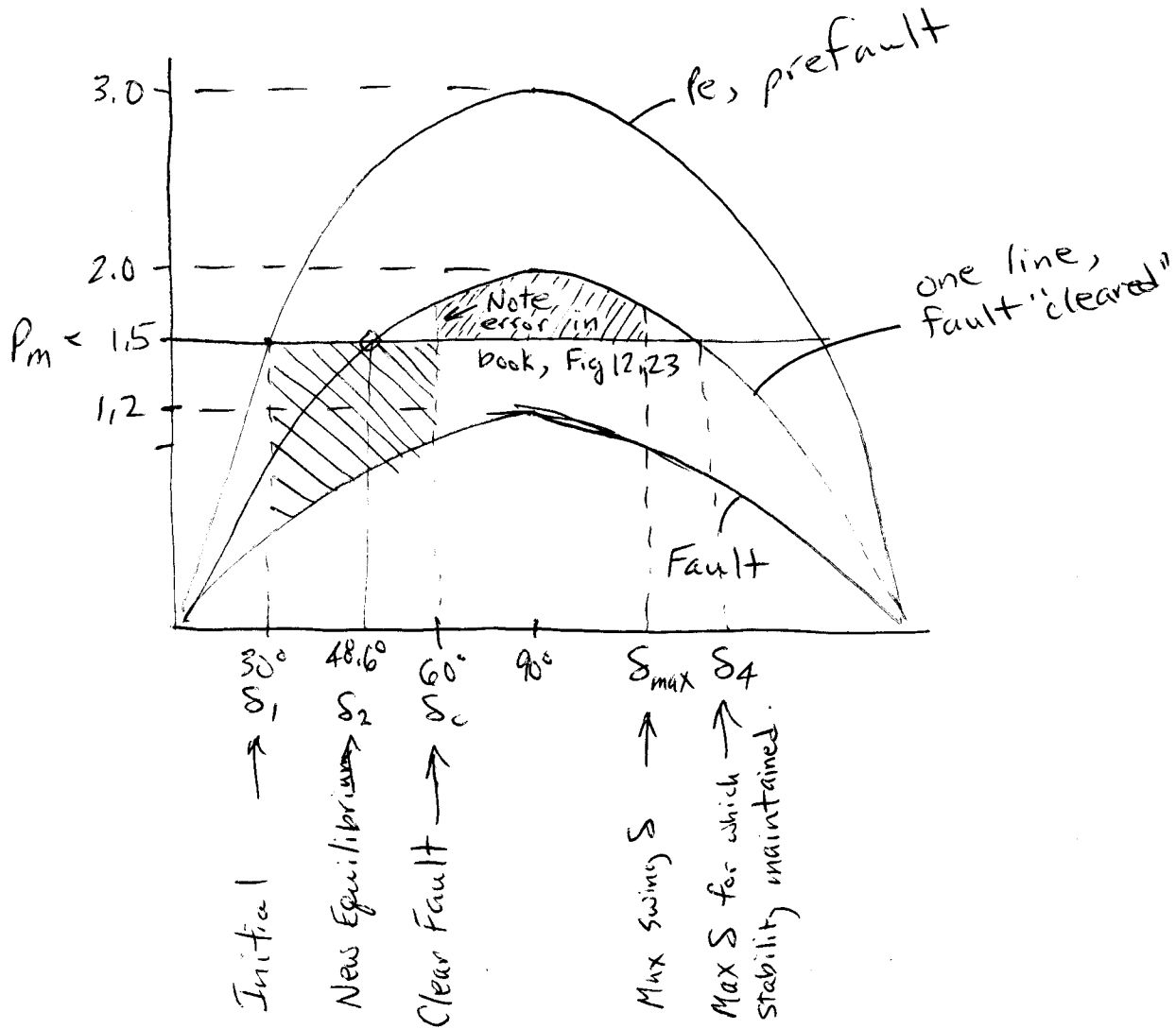
$$\therefore P_{e_{max}}^{Fault} = \frac{(1.2)(0.333)}{0.2 + 0.1333} = 1.2$$

Note: $P_m = 1.5$ Conclusion?

Impossible to maintain stability, if fault persists.

If fault can be cleared,

$$P_{e_{max}} = \frac{(1.2)(1)}{1.2 + 1.4} = 2.0 \quad (\text{from before, one line})$$



The accelerating area, A_1 , is

$$\int_{S_1}^{S_c} (P_m - P_e) dS = \int_{.524}^{1.047} (1.5 - 1.2 \sin S) dS$$

$$= 1.5 S + 1.2 \cos S \Big|_{.524}^{1.047} = 0.346$$

How far will δ increase after fault is cleared?

$$|A_2| = |A_1| \therefore \int_{\delta_{max}}^{1.047} (1.5 - 2.0 \sin \delta) d\delta = 0.346$$

$$1.5\delta + 2.0 \cos \delta \Big|_{\delta_{max}}^{1.047} = 0.346$$

$$1.5\delta_{max} + 2 \cos \delta_{max} = 0.346$$

$$\Rightarrow \delta_{max} = 1.848 \text{ rad} = 105.9^\circ$$

Max it could "swing" is out to $131.4^\circ = \delta_f$

Question: What is critical clearing angle?
 This is largest angle that fault could be cleared and stability still maintained.
 Call this δ_{cc} .

Ex 12.5

$$\int_{0.524}^{\delta_{cc}} \overbrace{(1.5 - 1.2 \sin \delta)}^{\text{faulted}} d\delta = \int_{2.293}^{\delta_{cc}} (1.5 - 2 \sin \delta) d\delta$$

Solving,

$$1.5\delta + 1.2 \cos \delta \Big|_{0.524}^{\delta_{cc}} = 1.5\delta + 2 \cos \delta \Big|_{2.293}^{\delta_{cc}}$$

$$\underline{\underline{\delta_{cc} = 1.196 \text{ rad} = 68.6^\circ}}$$

How long do we have to clear?

Not so simple, since the accelerating torque varies ~~e~~ with δ . Some key relationships:

$$P_{acc} = P_m - P_e = P_m - 1.2 \sin \delta$$

P_{acc} gradually decreases as δ increases to δ_{cc}

$$T_{acc} = (P_m - 1.2 \sin \delta) \omega_{\text{mech}}(\delta)$$

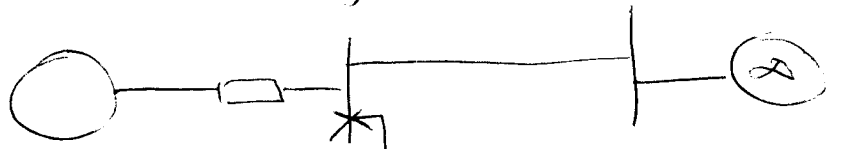
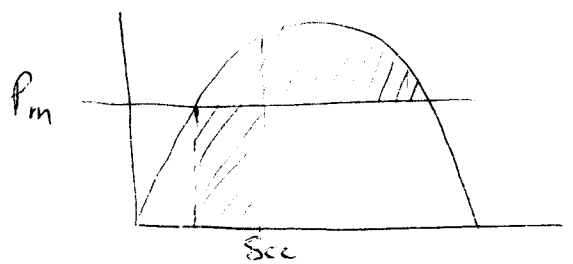
$$\omega = \omega_0 + \int_0^{t_{cc}} \alpha(\delta) dt \quad \leftarrow \quad \omega(t) = \omega_0 + \alpha t$$

From swing equation,

~~Equation~~ $\omega_{cc}^2 - \omega_1^2 = \frac{2\pi f}{H} \int_{\delta_1}^{\delta_{cc}} (P_m - P_e) d\delta$?

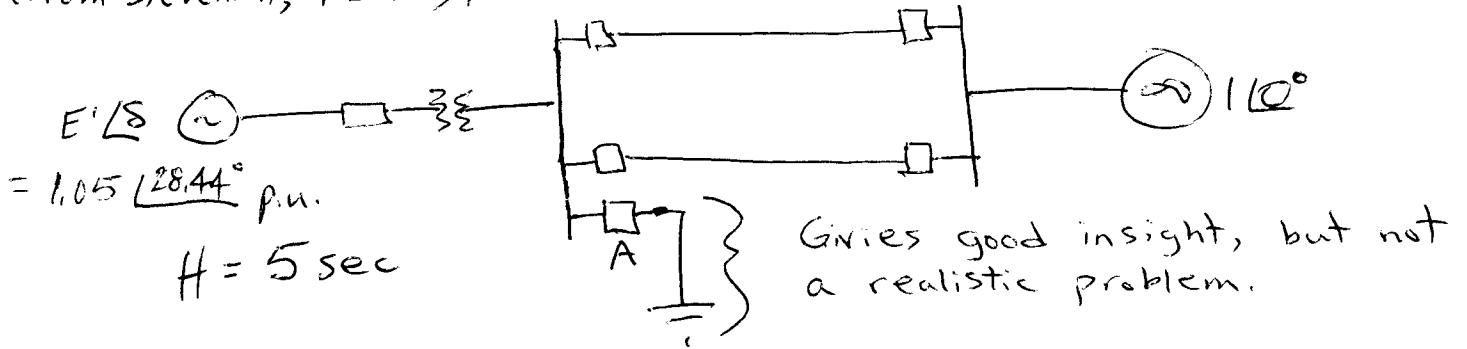
Should be able to get closed-form expression for $t_{critical}$. Usually this is done trial & error, in available texts anyway.

Easier case:

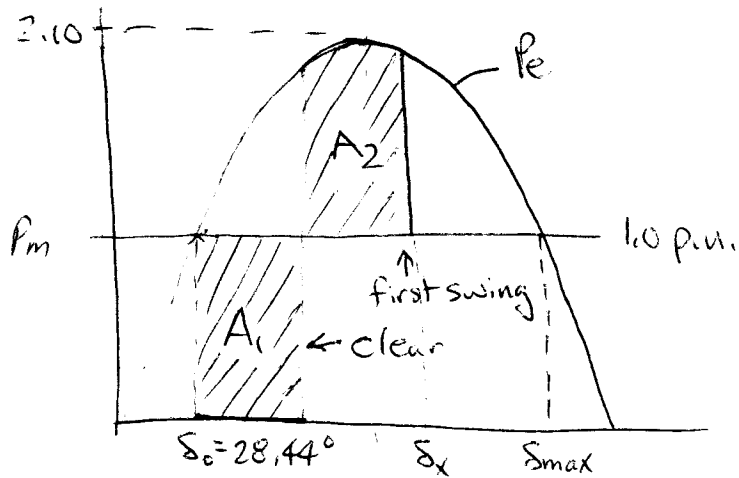


$P_e = 0$ during fault!

More on Equal Area Criterion: Fault on Gen Bus
 (From Stevenson, 4th Ed).



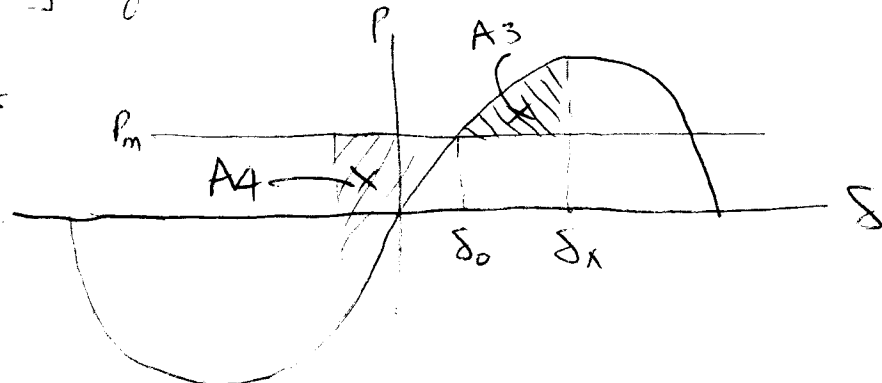
Assume a bus fault occurs (Breaker A is closed).
 Prefault P_{max} is 2.10 p.u.



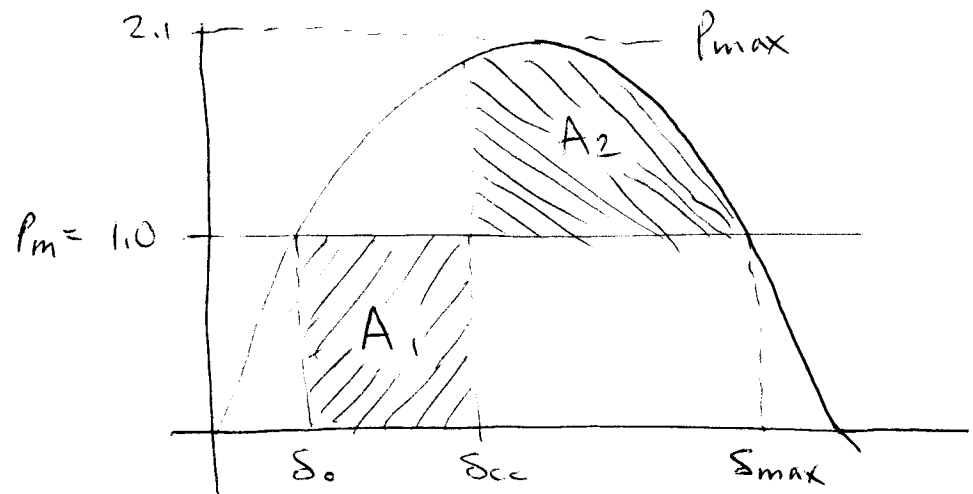
When fault occurs in this case, $P_e = 0$. $(P_m - P_e) = P_m$
 P_e remains at zero until fault is cleared (open A).
 δ will continue to increase until $A_2 = A_1$.

After clearing, equilibrium is still at δ_0

Now, δ swings about δ_0 , with
 $A_3 = A_4$



What is maximum δ_x for which stability can be maintained?



$$\underbrace{\int_{\delta_0}^{\delta_{cc}} (P_m - P_e) d\delta}_{A_1} = \underbrace{\int_{\delta_{max}}^{\delta_{cc}} (P_m - P_e) d\delta}_{A_2}$$

$$P_m \delta \Big|_{\delta_0}^{\delta_{cc}} = P_m \delta + P_{max} \cos \delta \Big|_{\delta_{max}}^{\delta_{cc}}$$

$$P_m \delta_{cc} - P_m \delta_0 = \cancel{P_m \delta_{cc}} - P_m \delta_{max} + P_{max} \cos \delta_{cc} - P_{max} \cos \delta_{max}$$

$$P_m (\delta_{max} - \delta_0) = P_{max} (\cos \delta_{cc} - \cos \delta_{max})$$

$$\cos \delta_{cc} = \frac{P_m}{P_{max}} (\delta_{max} - \delta_0) + \cos \delta_{max}$$

but $\delta_{max} = \pi - \delta_0$, $P_m = P_{max} \sin \delta_0$

$$\delta_{cc} = \cos^{-1} \left[(\pi - 2\delta_0) \sin \delta_0 - \cos \delta_0 \right] \text{ only for bus fault at gen!}$$

For this case, it is possible to directly calculate t_{cc} , the time before which the fault must be removed.

$P_A = P_m$, since P_e is zero in this case. Therefore, the accelerating Torque and the acceleration are roughly constant.

From swing equation, $\alpha = \frac{d^2\delta}{dt^2} = \frac{\pi f}{H} (P_m - P_e)$

$$\therefore \alpha = \frac{\pi f}{H} P_m \frac{\text{rad}}{\text{sec}^2}$$

$$\omega(t) = \omega_0 + \alpha t = \frac{\pi f P_m t}{H}$$

ω_0 (w ref = ω_s)

$$\delta(t) = \delta_0 + \int_0^t \omega(t) dt = \delta_0 + \frac{\pi f}{2H} t^2$$

$$\therefore \delta_{cc} = \delta_0 + \frac{\pi f}{2H} t_{cc}^2$$

$$\Rightarrow \boxed{t_{cc} = \sqrt{\frac{2H(\delta_{cc} - \delta_0)}{\pi f P_m}} \quad \begin{array}{l} \text{only for bus} \\ \text{fault at gen!} \\ (P_e = 0) \end{array}}$$

For previous problem, $\delta_0 = 28.44^\circ = 0.496 \text{ rad}$

$$\delta_{cc} = \cos^{-1} \left[(\pi - 2 \times 0.496) \sin 0.496 - \cos 0.496 \right] = 1.426 = \underline{\underline{81.70^\circ}}$$

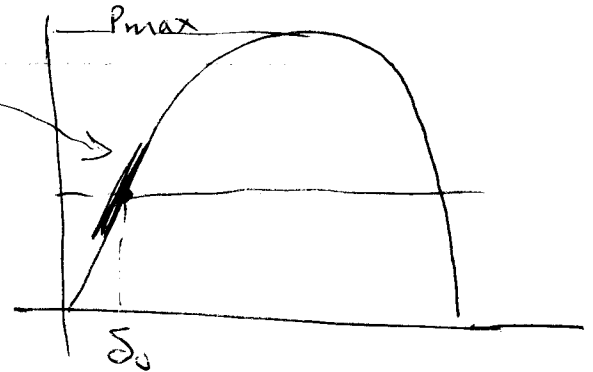
$$t_{cc} = \sqrt{\frac{2(5)(1.426 - 0.496)}{(60 \times \pi)(1.10)}} = \underline{\underline{0.222 \text{ sec}}}$$

Natural Frequency of Oscillation -

$$\omega_n = \sqrt{\frac{\omega_s P_{\max} \cos \delta}{2H}} \quad (\text{For small disturbances.})$$

$S_p = P_{\max} \cos \delta$ is the "synchronizing power coefficient". It is the slope of $P_e = P_{\max} \sin \delta$ at $\delta = \delta_0$.

$$f_n = \frac{\omega_n}{2\pi} = \frac{1}{2\pi} \sqrt{\frac{\omega_s P_{\max} \cos \delta}{2H}} \quad P_m$$



Ex: $\delta_0 = 28.44^\circ$, $H = 5 \text{ sec}$, $P_{\max} = 2.10 \text{ p.u.}$

$$\omega_n = \sqrt{\frac{(377)(2.10 \cos 28.44^\circ)}{2(5)}} = 8.343 \frac{\text{elec rad}}{\text{sec}}$$

$$f_n = \underline{\underline{1.33 \text{ Hz}}}$$

- Small perturbations in load (normal occurrence throughout day) will cause low-freq machine oscillations. Usually on order of $\sim 1 \text{ Hz}$. They are quickly damped by prime mover, generator and system load. Note that R of transmission system has virtually no damping effect.

12.8

Solution of Swing Equation:

For practical cases, critical clearing time cannot be calculated with equal area criterion. Engineers must know how much time they have to clear the fault - the fault must be detected, a trip signal sent to the CB, and then breaker must operate. t_{cc} must be known before the CB & protective relaying can be specified and before coordinating the relaying.

If we can solve for $\delta(t)$, then we can determine t_{cc} . The approach is typically to use trial & error, clearing fault at different times.

An analog computer implementation is given in book.

$\frac{\pi f}{H} (P_m - P_e)$ is integrated. (Note inverting integrator).
gain

basically:

$$\alpha(t) = \frac{d^2 \delta(t)}{dt^2} = \frac{\pi f}{H} (P_m - \underbrace{P_{\max} \sin(\delta(t))}_{P_e})$$

$$\omega(t) = \omega_0 + \int \alpha(t) dt$$

$$\delta(t) = \delta_0 + \int \omega(t) dt$$

Note that P_{\max} changes with fault, clear, reclose, ~~etc~~ line switching.

The book sets up a problem using Runge-Kutta numerical integration. This is more typical of the modern solution approach on computer.

Results for the previous example, where $\delta_{cc} = 68.6^\circ$, are given in Figs 12.25 & 12.26.

→ In these programs, t_{cc} is specified and simulation is run to see if stability is maintained.

** Best guess/judgement is that if ~~first pos/neg~~ first negative fluctuation in δ is \leq to first pos fluctuation then system will be stable.
i.e. if $(\delta_{\max} - \delta_0) \geq (\delta_0 - \delta_{\min}) \Rightarrow$ stable.

For this simple system (two machines), δ_{cc} could be found by equal-area. t_{cc} could be found simply by running simulation and noting the value of t when $\delta = \delta_{cc}$.

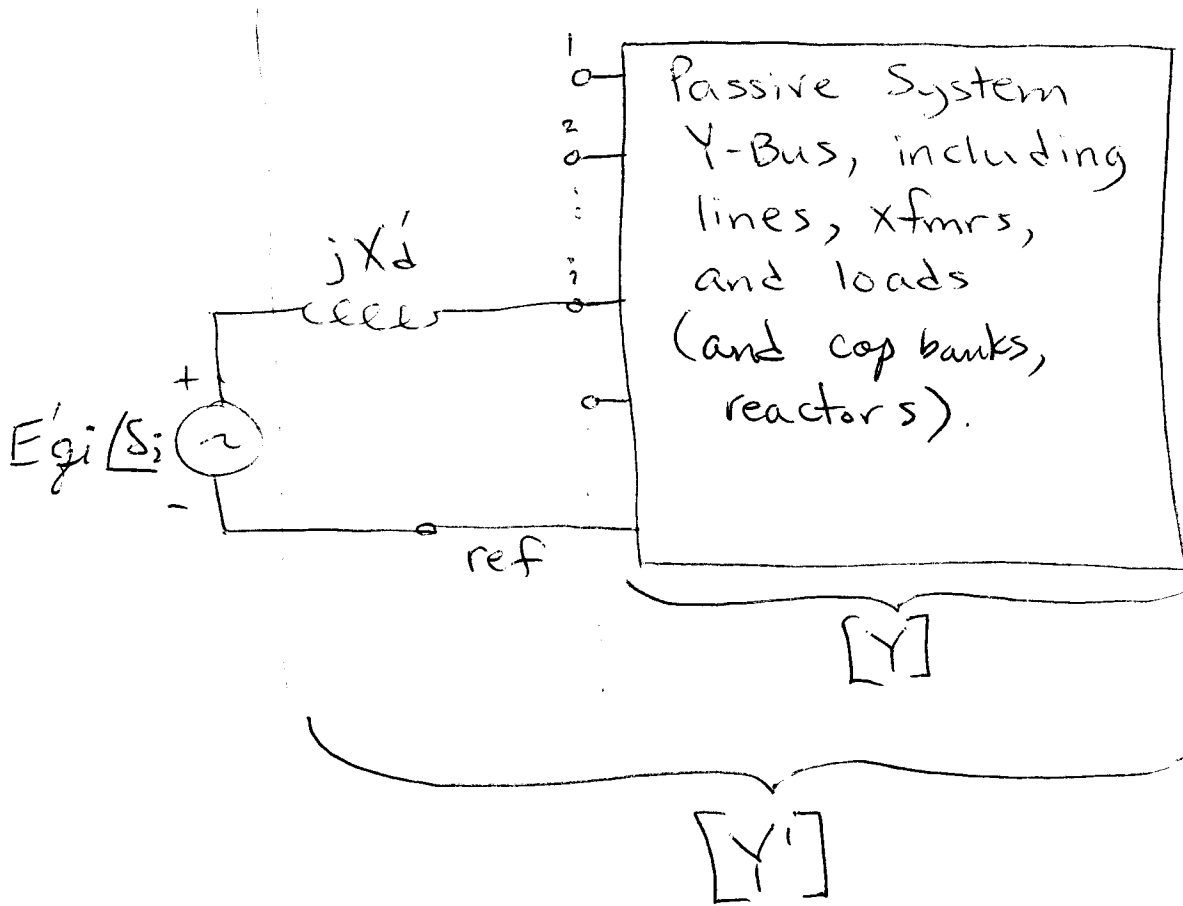
What if there are more than 2 machines?

Before going on,

- Do you understand swing equation?
- Do you understand [Y] system description?
- Do you have equal area method figured out?

12.9 - Multimachine Systems

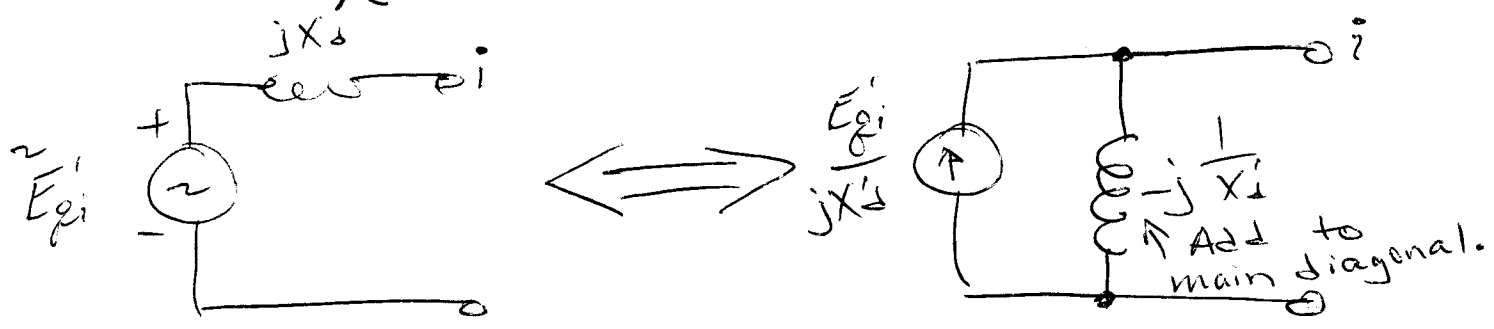
A network approach must be used where more than 2 machines are to be simulated.



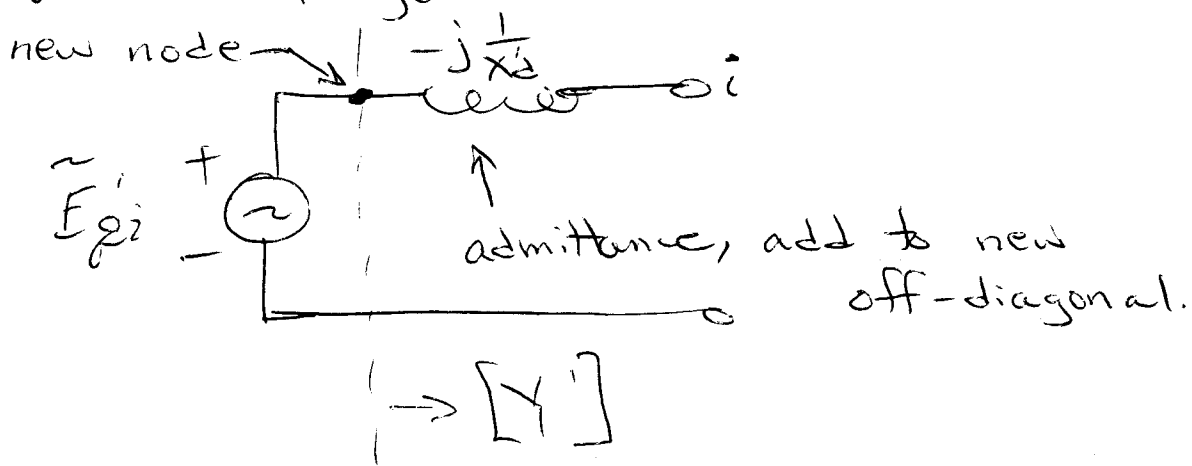
- Pre-~~transient~~ transient bus voltages typically obtained from load flow program.
- All buses not connected to a generator ~~are~~ have no current injected.

$$\begin{matrix}
 \left[\begin{matrix} I_1 \\ I_2 \\ \vdots \\ I_n \\ I_{n+1} \\ \vdots \\ I_{n+m} \end{matrix} \right] = [Y'] \left[\begin{matrix} E_1 \\ E_2 \\ \vdots \\ E_n \\ V_{n+1} \\ \vdots \\ V_{n+m} \end{matrix} \right] \\
 \left. \begin{matrix} \text{All} \\ \text{zero} \end{matrix} \right\} \left[\begin{matrix} I_1 \\ I_2 \\ \vdots \\ I_n \\ I_{n+1} \\ \vdots \\ I_{n+m} \end{matrix} \right] & \left. \begin{matrix} \text{n generators} \\ \text{m buses w/o generators,} \\ \text{Voltage due to } I_1 \rightarrow I_n \end{matrix} \right\} \left[\begin{matrix} E_1 \\ E_2 \\ \vdots \\ E_n \\ V_{n+1} \\ \vdots \\ V_{n+m} \end{matrix} \right]
 \end{matrix}$$

$[Y']$ is formed by adding Norton admittance of generators to main diagonal of $[Y]$. (No extra nodes added to network).



In this book, author adds an extra node at each generator bus:



Therefore, end up with $2n + m$ nodes in system. We will follow author's method, even though not the best in terms of computer application.

Nodes not connected to a generator are typically eliminated, using matrix partitioning.

$[Y]$ is partitioned:

$$\begin{array}{c|c} \begin{array}{c} 1 \rightarrow n \\ \downarrow \\ n \end{array} & \begin{array}{c} [Y_{AA}] \quad [Y_{AB}] \\ \hline [Y_{BA}] \quad [Y_{BB}] \end{array} \\ \begin{array}{c} n+1 \\ \downarrow \\ n+m \end{array} & \end{array}$$

$$[Y_{red}] = [Y_{AA}] - [Y_{AB}][Y_{BB}]^{-1}[Y_{BA}] = n \times n$$

See derivation, p. 533

Ex: Go thru Ex. 12.7

Effect of loads: Loads are added to main diagonal term of $[Y]$

Ex: see example 12.8

Calculating P_{max} for various ~~conditions~~ conditions is done using $[Y]$ for each of the operating states.

Ex: IF $[Y] = \begin{bmatrix} -j2.5 & j2.5 \\ j2.5 & -j2.5 \end{bmatrix}$ prefault

$$\begin{array}{ll} E_1 = E_1' = 1.2 \text{ p.u.} & \delta_1 = \delta \\ E_2 = 1.0 \text{ p.u.} & \delta_2 = 0 \text{ (infinite bus)} \end{array} \quad \begin{array}{l} g_1 \angle \delta_1 = 2.5 \angle -90^\circ \\ g_2 \angle \delta_2 = 2.5 \angle +90^\circ \end{array}$$

$$\begin{aligned} P_{e1} &= \sum_{j=1}^2 E_i E_j y_{ij} \cos(\delta_i - \delta_j - \alpha_{ij}) \\ &= (1.2)^2 (2.5) \cos(\delta_1 - \delta_1 - (-90)) + (1.2)(1.0)(2.5) \cos(\delta_1 - 0 - 90) \\ &= 3.0 \sin \delta \Rightarrow P_{max} = 3.0 \end{aligned}$$

Using same method,

$$P_{e\max} = 1.2 \quad \text{during fault.}$$

$$P_{e\max} = 2.0 \quad \text{when cleared.}$$

Each machine's swing equation is solved separately, although δ depends on other machines.

$$P_{mi} - P_{ei} = \frac{H_i}{\pi f} \left(\frac{d^2 \delta_i}{dt^2} + \frac{d\omega_{ref}}{dt} \right) \quad i = 1, 2, \dots, n$$

Caution: use ~~same~~ system base for all H and P values.

Usually pick infinite bus machine for ω_{ref}

$$P_{mi} - P_{ei} = \frac{H_i}{\pi f} \left(\frac{d^2 \delta_i}{dt^2} \right) \quad i = 1, 2, \dots, n$$

Reformulate as

$$\frac{d\delta_i}{dt} = \omega_i$$

$$\frac{d\omega_i}{dt} = \frac{\pi f}{H_i} \left(P_{mi} - \underbrace{\sum_{j=1}^n E_i E_j y_{ij} \cos(\delta_i - \delta_j - \alpha_{ij})}_{P_{ei}} \right)$$

Initial conditions: $\omega_i = 0$ for all machines $\omega_i(0)$
 $\delta_{0i} = \delta_i(0)$

$$\frac{ds(t)}{dt} = f = \omega(t)$$

$$\frac{d\omega(t)}{dt} = g = \alpha(t) = \frac{\pi f}{H} (P_m - P_e)$$

$$s(0) = s_0 = \sin^{-1}\left(\frac{P_m}{P_{max}}\right) \quad \text{INITIATE}$$

$$\omega(0) = 0 \quad \alpha(0) = \text{CALCULATE}$$

$$f(s,t) = \omega_{avg} = \omega(t)$$

$$g(s,t) \text{ (alpha)} = \frac{\pi f}{H} [P_m - P_{max} \sin S(t)]$$

~~$$K1 = \omega_{avg} * dt$$

$$L1 = \alpha * dt$$

$$K2 = \omega_{avg} * dt$$

$$L2 = \alpha \left(s + \frac{K1}{2}\right) * dt$$

$$K3 = \omega_{avg} * dt$$

$$L3 = \alpha \left(s + \frac{L2}{2}\right) * dt$$

$$K4 = \omega_{avg} * dt$$

$$L4 = \alpha \left(s + \frac{L3}{2}\right) * dt$$~~

$$K1 = \omega_{avg} * dt = ds$$

$$L1 = \alpha^{(s)} * dt$$

$$K2 = \omega_{avg} * dt$$

$$L2 = \alpha \left(s + \frac{K1}{2}\right) * dt$$

$$K3 = \omega_{avg} * dt$$

$$L3 = \alpha \left(s + \frac{K2}{2}\right) * dt$$

$$K4 = \omega_{avg} * dt$$

$$L4 = \alpha \left(s + \frac{K3}{2}\right) * dt$$

$$D\omega_{avg} = \omega_{avg} * dt$$

$$D\Delta = (L1 + 2 * L2 + 2 * L3 + L4) / 6.0$$

~~FUNCTION ALPHA (DECTA, PM, PMAX)~~

$$\omega_{avg} = \omega_{avg} + D\omega_{avg}$$

$$\Delta = \Delta + D\Delta$$

$$\omega_{avg} (0:1000)$$

$$\Delta (0:1000)$$

$$\alpha (0:1000)$$

LECTURE 22

23

MULTI-MACHINE STUDIES

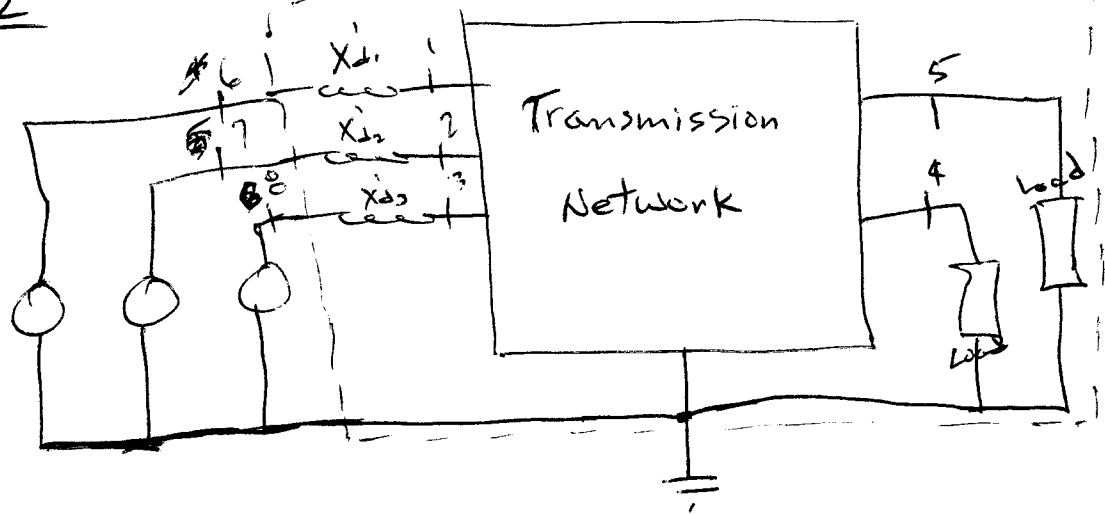
(Can't use equal-area if > 2 machines)

- Use load flow to establish initial conditions.
- Convert loads to Admittances
(Can be constant admittance, or voltage dependent.)

ex:

$$Y_{LS} = \frac{P_L - jQ_L}{|V_L|^2} \quad \text{then add to } Y_{SS}$$

Fig 14.2



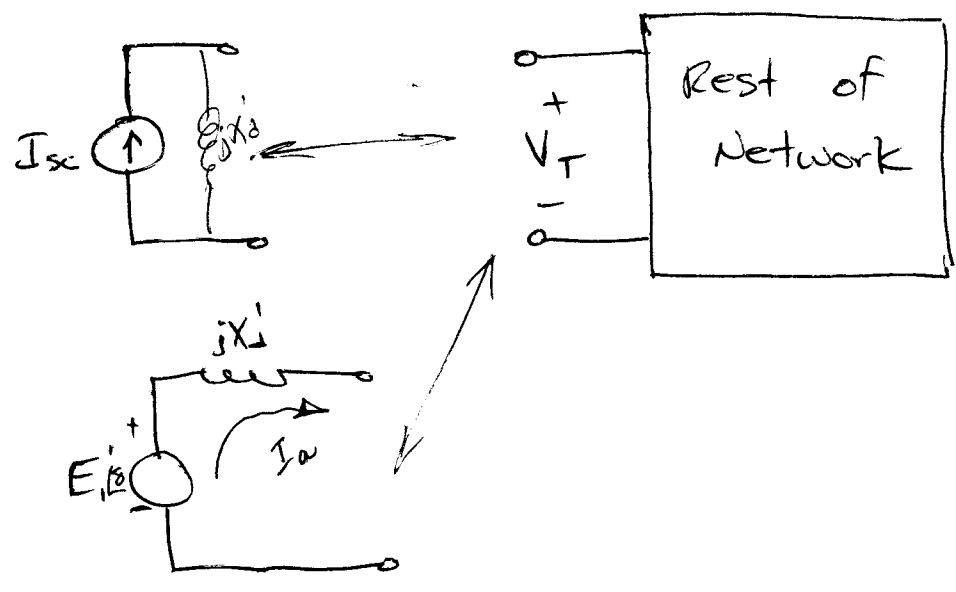
- Two ways to absorb X_d'
- Make new bus
 - Use Norton Equiv

Electrical power:
Add buses:

$$P_{e1} = |E_1'|^2 G_{11} + |E_1'| |E_2'| |Y_{12}| \cos(\delta_{12} - \theta_{12}) + |E_1'| |E_3'| |Y_{13}| \cos(\delta_{12} - \theta_{12})$$

So do summation

OR: use Norton Equiv
 For each Mach:



~~you know~~

and power flow

1) Use V_T to find $|E'| = \text{constant}$.
 (From load flow) and S_o & I_o

2) Switch system (Fault, etc)

$$P_e = E_1' \cdot I_a^* = \frac{E_1' [(E_1' - V_T)]^*}{jX_d'}$$

Modified Euler Method - ^{Adapted} From Stagg & El-Abiad 3

- 1) Use V_T and power flow from Load flow to find $|E'|_0, \delta_0, I_{a0}, P_m$
 $P_m = \text{Re}\{E'_0 I_{a0}^*\} = \text{constant}$
- 2) Perform system switching (Fault) etc
(Modify Y-Bus)
- 3) Use $I_{sc(0)} = \frac{E'_0}{jX'_d}$ and solve $V_{(0)} = Y^{-1} I_{sc(0)}$ for all machines.
- 4) Calculate new machine currents based on $\frac{E'_0 - V}{jX'_d} = I_a(0^+)$

- 5) Calculate machine electrical Power
 $P_e = \text{Re}\{E'_0 I_a^*(0^+)\}$

$$\begin{aligned} 6) \quad \omega(t+\Delta t) &= \omega(t) + \left[\frac{d\omega}{dt}(\omega(t)) \right] \Delta t \quad \text{from swing eqn.} \\ &= \left(\omega(t) + \left[\frac{\pi f}{H} (P_m - P_e) \right] \Delta t \right) \text{est.} \end{aligned}$$

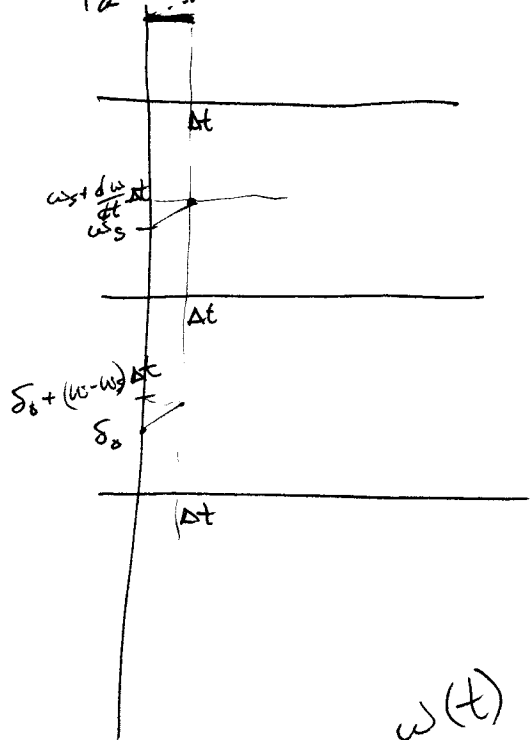
$$\begin{aligned} \delta(t+\Delta t) &= \delta(t) + \frac{d\delta(t)}{dt} \Delta t \\ &= \left(\delta(t) + [\omega(t) - \omega_s] \Delta t \right) \text{est.} \end{aligned}$$

- 7) Calculate ~~new~~ est voltages behind $jX'_d \Rightarrow |E'|_{\delta(t+\Delta t)}$

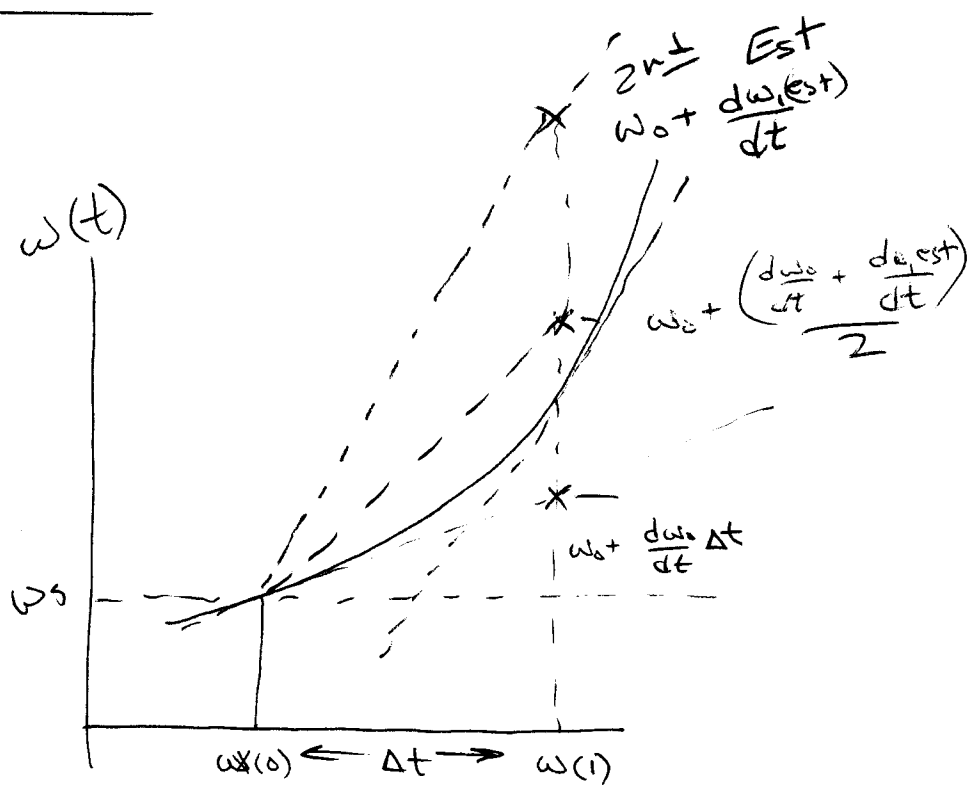
- 8) Use $|E'|_{\delta(t+\Delta t)}$ (est) to determine $V(t+\Delta t) = Y^{-1} I_{sc}(t+\Delta t)$

- 9) Calculate $P_e = \text{Re}\{E'_{(t+\Delta t)} I_{a(t+\Delta t)}^*\}$

$$P_a = \frac{H}{\pi f} \frac{d\delta^0}{dt^2} = P_m - P_e$$

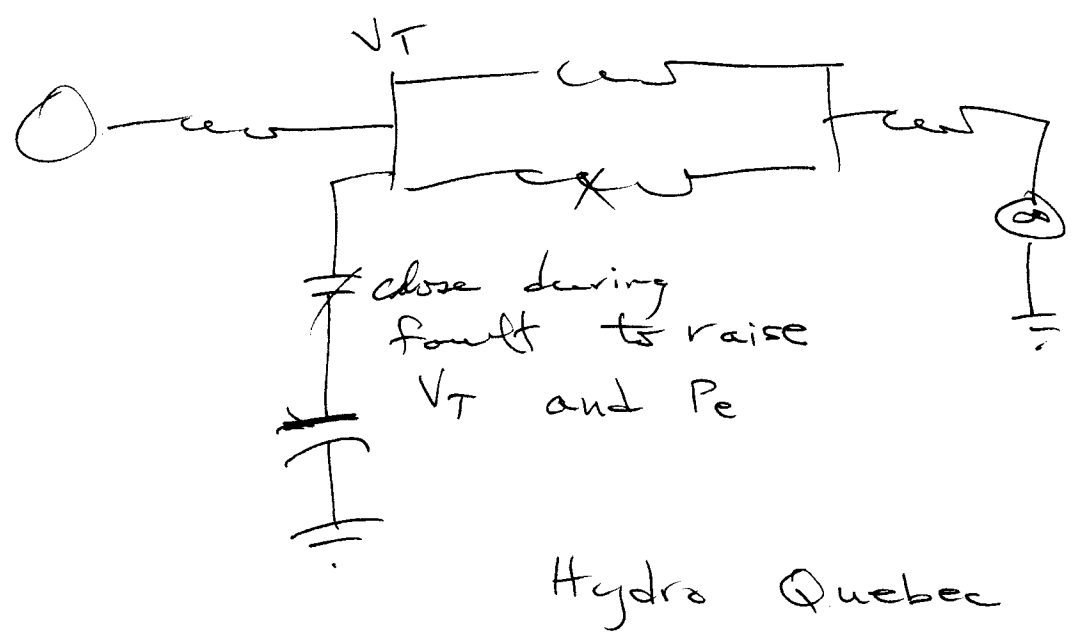


MODIFIED
EULER



Factors Affecting Stability

- Gov
- Exciter
- PSS (Power System Stabilizer)
works on Δf from f_s changes V_{int}
faster than Exciter ΔV controls
- STATIC VARZ COMP



Fast Valve

Braking Resist