

Stream Function: Section 4.2. also Middleman text. p366-371

The Stream Function and solutions in terms of the Stream Function are very useful in solving 2-dimensional steady flows of Newtonian fluids. For 2-dim. flows, we have 3 dependent variables (2 velocity components + pressure) to solve for. By using the Stream Function, we are able to reduce this to just one dependent variable!

The Stream Function also has a geometric interpretation, as we will demonstrate next.

In Cartesian coordinates and for 2-dim. flow, we have 2 velocity components which can be expressed in terms of the Stream Function,

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Definition of ψ

$$v_x = \frac{\partial \psi}{\partial y} (-) \quad v_y = \frac{\partial \psi}{\partial x} (+)$$

where $\psi \equiv$ stream function. $\psi = \psi(x, y)$ varies in x and y directions for 2-dim. flow. A differential change in ψ can be related to dx and dy,

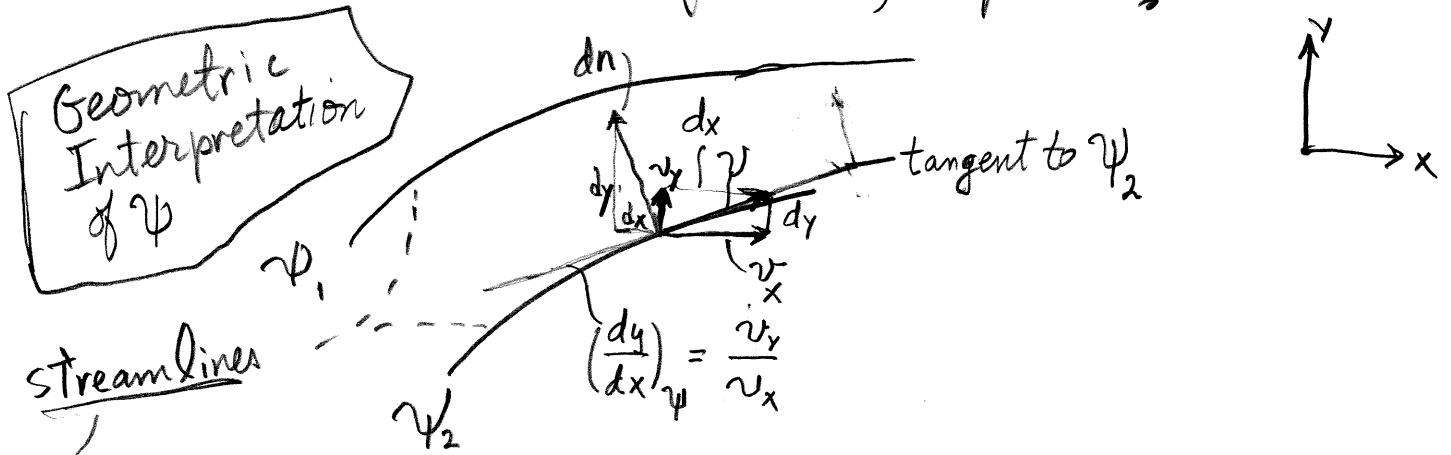
$$d\psi = \frac{\partial \psi}{\partial x} dx + \frac{\partial \psi}{\partial y} dy$$

$$\left. \begin{array}{l} \\ \end{array} \right\} = v_y dx - v_x dy \quad (\text{using definitions of } \psi \text{ from above.})$$

Along a line of constant ψ , $d\psi = 0$

$$\left[\frac{dy}{dx} \right]_{\psi} = \frac{v_y}{v_x}$$

In other words, the path that would be taken by a fluid element through space $[dy/dx]_{\psi}$ at a constant value of ψ ($d\psi=0$) would be aligned with the fluid velocity (v_y/v_x) components!



Thus, the flow path within the fluid are along paths of constant ψ , thus the Stream Function is valuable for flow visualization.

Another useful property of ψ is its relationship to the mass flow rate.

$$\dot{m} = \int_{\psi_1}^{\psi_2} \rho v \, dn$$

where
 $\dot{m} = \frac{\text{mass}}{\text{time} \cdot \text{unit width in } z\text{-direction}}$

$\rho = \text{fluid density } (\frac{\text{mass}}{\text{volume}})$

$v = \text{velocity vector (magnitude only)}$
 $= \sqrt{v_x^2 + v_y^2}$

$dn = \text{a differential distance } \perp \text{ to a stream line (const. } \psi \text{)}$.

To complete this analysis, we need to relate $d\psi$ to dn and to v .

We note that $\frac{dx}{dn} = +\frac{v_y}{v}$ $\frac{dy}{dn} = \frac{v_x}{v}$

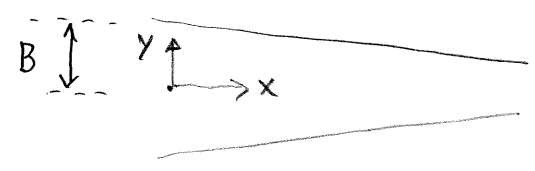
$$\begin{aligned}
 \frac{d\psi}{dn} &= \left(\frac{\partial\psi}{\partial x}\right) \frac{dx}{dn} + \left(\frac{\partial\psi}{\partial y}\right) \frac{dy}{dn} \\
 &= +v_y \frac{dx}{dn} + v_x \frac{dy}{dn} \\
 &= +v_y \left(\frac{v_y}{v}\right) + v_x \left(\frac{v_x}{v}\right) \\
 &= \frac{v_y^2 + v_x^2}{v} = \frac{v^2}{v} = \boxed{v = \frac{\partial\psi}{\partial n}}
 \end{aligned}$$

$$\dot{m} = \int_{\psi_1}^{\psi_2} \rho v dn = \int_{\psi_1}^{\psi_2} \rho \frac{\partial\psi}{\partial n} dn = \int_{\psi_1}^{\psi_2} \rho d\psi = \boxed{\rho(\psi_2 - \psi_1)}$$

Thus, the mass flow rate between streamlines is proportional to the difference between them. Thus, if ψ is displayed at equal $\Delta\psi$ between them, regions of the greatest mass flow rate will have streamlines closest together.

Example: Laminar Flow Between Parallel Plates:

The general 2-dim. flow case is when the plates are not exactly parallel.



$$\begin{aligned}
 v_x &= v_x(x, y) \\
 v_y &= v_y(x, y)
 \end{aligned}$$

The Continuity Eqn and Navier-Stokes Eqns. are.

$$\frac{\partial v_x}{\partial x} + \frac{\partial v_y}{\partial y} = 0$$

3 eqns.
3 unknowns:
 v_x, v_y, P

Continuity.

$$\rho \left(v_x \frac{\partial v_x}{\partial x} + v_y \frac{\partial v_x}{\partial y} \right) = -\frac{\partial P}{\partial x} + \mu \left(\frac{\partial^2 v_x}{\partial x^2} + \frac{\partial^2 v_x}{\partial y^2} \right) \quad \text{x-component}$$

$$\rho \left(v_x \frac{\partial v_y}{\partial x} + v_y \frac{\partial v_y}{\partial y} \right) = -\frac{\partial P}{\partial y} + \mu \left(\frac{\partial^2 v_y}{\partial x^2} + \frac{\partial^2 v_y}{\partial y^2} \right) \quad \text{y-component}$$

In terms of ψ . the Continuity Eqn. becomes.

$$-\frac{\partial^2 \psi}{\partial x \partial y} + \frac{\partial^2 \psi}{\partial y \partial x} = 0 \quad \text{which is satisfied for any } \psi(x,y)!$$

Thus we lose 1 eqn, but we have only 2 unknowns, ψ, P .

We eliminate P by,

subst. definition of ψ into x- and y-component eqns.

$\frac{\partial}{\partial y}$ x-comp. eqn. }
 $\frac{\partial}{\partial x}$ y-component eqn. } subt. y-comp. eqn. from x-comp. eqn., $\frac{\partial^2 P}{\partial x \partial y}$ terms cancel.

$$\boxed{\frac{\partial \psi}{\partial x} \frac{\partial (\nabla^2 \psi)}{\partial y} - \frac{\partial \psi}{\partial y} \frac{\partial (\nabla^2 \psi)}{\partial x} = \nu \nabla^4 \psi}$$

$$\nabla^2 \psi \equiv \frac{\partial^2 \psi}{\partial x^2} + \frac{\partial^2 \psi}{\partial y^2}$$

$$\nabla^4 \psi \equiv \frac{\partial^4 \psi}{\partial x^4} + 2 \frac{\partial^4 \psi}{\partial x^2 \partial y^2} + \frac{\partial^4 \psi}{\partial y^4}$$

see Table 4. for ψ eqns in other coord. systems.