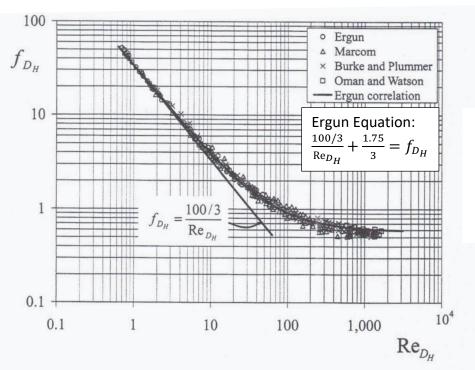
Source: F. A. Morrison, An Introduction to Fluid Mechanics, Cambridge, 2013



 $f_{D_H} \equiv \left(\frac{\Delta p}{L}\right) \frac{D_H \varepsilon^2}{2\rho v_0^2}$ 

$$\operatorname{Re}_{D_H} \equiv \frac{\rho(v_0/\varepsilon)D_H}{u}$$

Figure 7.40

Data on friction factor versus Reynolds number at low Reynolds number validate the hypothesis that  $t_{D_H} \operatorname{Re}_{D_H}$  is constant for slow flow through packed beds. The constant  $f_{D_H} \operatorname{Re}_{D_H}$  is found to be 100/3 = 33.33. Above  $\operatorname{Re}_{D_H} = 10$ , the data deviate from the hydraulic-diameter result, following instead  $f_{D_H} = \operatorname{constant} = 1.75/3$ , which is the result expected for flow in very rough pipe (compare to Figure 7.22 for large  $\varepsilon/D$  and large Re). Data are from reference [45].

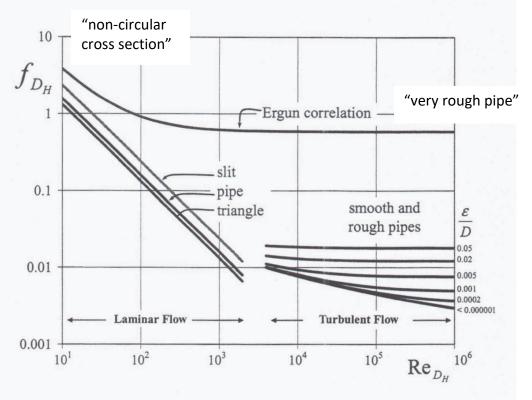


Figure 7.41

We compare the packed-bed result (i.e., Ergun correlation) with the friction-factor/Reynolds-number relationship for flows in other conduits. At low Reynolds number, the Poiseuille number,  $Po = f_{D_H} Re_{D_H}$ , is constant for most cross-sectional shapes and for packed beds. At high Reynolds number and high roughness in pipe flow, the friction factor becomes constant with a value that increases with increasing roughness; packed beds at high Re also have  $f_{D_H} = constant$ . At intermediate Reynolds number, the observed behavior of packed beds is intermediate between these two extremes.