

## CM3120: Module 2

### Unsteady State Heat Transfer

- I. Introduction
- II. Unsteady Microscopic Energy Balance—(slash and burn)
- III. Unsteady Macroscopic Energy Balance
- IV. Dimensional Analysis (unsteady)—Biot number, Fourier number
- V. Low Biot number solutions—Lumped parameter analysis
- VI. Short Cut Solutions—(initial temperature  $T_0$ ; finite  $h$ ), Gurney and Lurie charts (as a function of position,  $m = 1/Bi$ , and  $Fo$ ); Heissler charts (center point only, as a function of  $m = 1/Bi$ , and  $Fo$ )
- VII. Full Analytical Solutions (stretch)

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## CM3120: Module 2

### Module 2 Lecture VI: Short Cut Solutions (Gurney and Lurie/Heisler Charts)



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[www.chem.mtu.edu/~fmorriso/cm3120/cm3120.html](http://www.chem.mtu.edu/~fmorriso/cm3120/cm3120.html)

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In a previous lecture, we found that **Dimensional Analysis** helped us to organize our “tool belt” for engineering problem solving.

For **Unsteady Heat Transfer** problems, we added two dimensionless numbers, the **Biot number** (*bee oh*)  $Bi$  and the **Fourier number**  $Fo$

**CM3120: Module 2**

**Dimensional Analysis**  
*For Unsteady State Heat Transfer*





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www.chem.mtu.edu/~fmorriso/cm3120/cm3120.html

**How can we organize our tool belt?**

**What is our usual strategy for complex phenomena?**

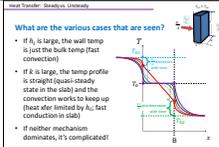
**Answer: Dimensional Analysis**

- ✓ Let's nondimensionalize the governing equations and BCs.
- ✓ Let's sort out the various unsteady cases.

Heat Transfer: Steady, Unsteady

What are the various cases that are seen?

- If  $h_c$  is large, the wall temp is just the bulk temp (flat convection)
- If  $h_c$  is large, the temp profile is straight (quasi-steady state in the slab) and the convection works to keep up (heat rate limited by  $h_c$ -fast conduction in slab)
- If neither mechanism dominates, it's complicated!



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Unsteady State Heat Transfer in a Body

**Two Additional Dimensionless Numbers**

**$Bi$  – Biot Number =  $\frac{hD}{k}$**

Quantifies the tradeoffs between the internal resistance  $D/k$  (due to conduction) and  $1/h$ , the resistance at the phase boundary (due to convection)

**$Fo$  – Fourier Number =  $\frac{\alpha t}{D^2}$**

Scales the time evolution of the temperature profile relative to the material's thermal properties,  $\alpha = k/\rho\hat{C}_p$  (thermal diffusion time).

**Dimensionless Numbers**

<p><small>momentum energy mass</small></p> <p><b>Re</b> – Reynolds = <math>\frac{\rho v D}{\mu} = \frac{v D}{\nu}</math></p> <p><b>Fr</b> – Froude = <math>\frac{v^2}{g D}</math></p> <p><b>Pe</b> – Péclet<sub>n</sub> = <math>\text{RePr} = \frac{\hat{C}_p \rho v D}{k} = \frac{v D}{\alpha}</math></p> <p><b>Pe</b> – Péclet<sub>m</sub> = <math>\text{ReSc} = \frac{v D}{D_{AB}}</math></p>	<p>These numbers from the governing equations tell us about the relative importance of the terms they precede in the microscopic balances (<b>scenario properties</b>).</p>
<p><b>Pr</b> – Prandtl = <math>\frac{\hat{C}_p \mu}{k} = \frac{\nu}{\alpha}</math></p> <p><b>Sc</b> – Schmidt = <math>\text{LePr} = \frac{\mu}{\rho D_{AB}} = \frac{\nu}{D_{AB}}</math></p> <p><b>Le</b> – Lewis = <math>\frac{\alpha}{D_{AB}}</math></p>	<p>These numbers compare the magnitudes of the diffusive transport coefficients <math>\nu, \alpha, D_{AB}</math> (<b>material properties</b>).</p>
<p><b>f</b> – Friction Factor = <math>\frac{\mathcal{F}_{drag}}{(\frac{\rho}{2} v^2) A_c}</math></p> <p><b>Nu</b> – Nusselt = <math>\frac{h D}{k}</math></p> <p><b>Sh</b> – Sherwood = <math>\frac{k_m D}{D_{AB}}</math></p>	<p>These numbers are defined to help us build transport data correlations based on the fewest number of grouped (dimensionless) variables (<b>scenario properties</b>).</p>

**We indicated that there are three ranges of Biot number to consider:**

At high Bi, the surface temperature equals the bulk temperature; heat transfer is limited by conduction in the body.

At moderate Bi, heat transfer is affected by both conduction in the body and the rate of heat transfer to the surface.

At low Bi, the temperature is uniform in a finite body; heat transfer is limited by rate of heat transfer to the surface ( $h$ ).

**Bi – Biot Number** =  $\frac{hD}{k}$

**Bi** =  $\frac{D_{char}/k}{1/h}$

**High Bi:**  
low  $k$ ,  
high  $h$

**Moderate Bi:**  
nether process dominates

**Low Bi:**  
high  $k$ ,  
low  $h$

**We have been exploring these ranges**

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**Summary of low Biot number scenarios:**

Unsteady State Heat Transfer: Ranges of Biot Number

**Lumped parameter analysis**  
(negligible internal resistance)

At low Bi, the temperature is uniform in a finite body; heat transfer is limited by rate of heat transfer to the surface ( $h$ ).

$$D_{char} \equiv \frac{\text{volume}}{\text{area}} = \frac{V_{sys}}{A_{sys}}$$

This is **always** the  $D_{char}$  we use for the Biot number in lumped parameter analysis. We use different  $D_{char}$  in other cases, however.

The test for applicability of the lumped parameter analysis is:

$$Bi_{LP} = \frac{hV_{sys}}{kA_{sys}}$$

$$Bi_{LP} = \frac{hV_{sys}}{kA_{sys}} < 0.1$$

**Low Bi:**  
high  $k$ ,  
low  $h$

**Two things** to remember about lumped parameter analysis:

1.  $D_{char} \equiv \frac{V}{A}$
2. Only valid for  $Bi_{LP} < 0.1$

Low Biot number  $\Rightarrow$  temperature is uniform in the body; resistance is all **external**; solve for  $T_{body}(t)$

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## Now: Moderate and High Biot number behavior

We indicated that there are three ranges of Biot number to consider:

At high Bi, the surface temperature equals the bulk temperature; heat transfer is limited by conduction in the body.

At moderate Bi, heat transfer is affected by both conduction in the body and the rate of heat transfer to the surface.

At low Bi, the temperature is uniform in a finite body; heat transfer is limited by rate of heat transfer to the surface ( $h$ ).

We now explore these ranges

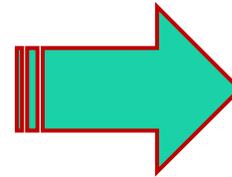
$$Bi - \text{Biot Number} = \frac{hD}{k}$$

$$Bi = \frac{D_{char}/k}{1/h}$$

**High Bi:**  
low  $k$ ,  
high  $h$

**Moderate Bi:**  
nether process dominates

**Low Bi:**  
high  $k$ ,  
low  $h$



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$$Bi - \text{Biot Number} = \frac{hD}{k}$$

$$Bi = \frac{D/k}{1/h}$$

Quantifies the tradeoffs between the resistance to heat flow (due to conductivity,  $D/k$ ) and the resistance to heat flow at the boundary ( $1/h$ )

When both processes affect the outcomes, the full solution may be necessary. For uniform starting temperatures, the solutions are published.

the body.

$$T = T(x, y, z, t)$$

hard BC

At moderate Bi, heat transfer is affected by both conduction in the body and the rate of heat transfer to the surface.

**Moderate Bi:**  
nether process dominates

At low Bi, the temperature is uniform in a finite body; heat transfer is limited by rate of heat transfer to the surface ( $h$ ).

**Low Bi:**  
high  $k$ ,  
low  $h$

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All solutions for the temperature *profile* (temperature *distribution*, temperature *field*,  $T(t, \underline{x})$ ) begin with the

**Microscopic Energy Balance**

**Bi – Biot Number =  $\frac{hD}{k}$**   
 Quantifies the tradeoffs between the resistance to heat flow (due to conductivity,  $D/k$ ) and the resistance to heat flow at the boundary ( $1/h$ )

$Bi = \frac{D/k}{1/h}$

When both processes affect the outcomes, the full solution may be necessary. For uniform starting temperatures, the solutions are published.

$T = T(x, y, z, t)$

At moderate Bi, heat transfer is affected by both conduction in the body and the rate of heat transfer to the surface. **Moderate Bi: neither process dominates**

At low Bi, the temperature is uniform in a finite body; heat transfer is limited by rate of heat transfer to the surface ( $h$ ). **Low Bi: high  $k$ , low  $h$**

1D Heat Transfer: Unsteady State

**Microscopic Energy Balance**

As for the derivation of the microscopic momentum balance, the microscopic energy balance is derived on an arbitrary volume,  $V$ , enclosed by a surface,  $S$ .

Gibbs notation:

$$\rho \hat{C}_p \left( \frac{\partial T}{\partial t} + \underline{v} \cdot \nabla T \right) = k \nabla^2 T + S$$

see handout for component notation

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1D Heat Transfer: Unsteady State

**Microscopic Energy Balance**

(General Energy Transport Equation)

$$\rho \hat{C}_p \left( \frac{\partial T}{\partial t} + \underline{v} \cdot \nabla T \right) = k \nabla^2 T + S$$

rate of change

convection

source (energy generated per unit volume per time)

conduction (all directions)

velocity must satisfy equation of motion, equation of continuity

see handout for component notation

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**Equation of energy** for Newtonian fluids of constant density,  $\rho$ , and thermal conductivity,  $k$ , with source term (source could be viscous dissipation, electrical energy, chemical energy, etc., with units of energy/(volume time)).

Source: R. B. Bird, W. E. Stewart, and E. N. Lightfoot, *Transport Processes*, Wiley, NY, 1960, page 319.

Gibbs notation (vector notation)

[www.chem.mtu.edu/~fmorriso/cm310/energy2013.pdf](http://www.chem.mtu.edu/~fmorriso/cm310/energy2013.pdf)

$$\left(\frac{\partial T}{\partial t} + \mathbf{v} \cdot \nabla T\right) = \frac{k}{\rho \hat{C}_p} \nabla^2 T + \frac{S}{\rho \hat{C}_p}$$

thermal diffusivity  $\alpha \equiv \frac{k}{\rho \hat{C}_p}$

Cartesian (xyz) coordinates:

$$\frac{\partial T}{\partial t} + v_x \frac{\partial T}{\partial x} + v_y \frac{\partial T}{\partial y} + v_z \frac{\partial T}{\partial z} = \frac{k}{\rho \hat{C}_p} \left( \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} + \frac{\partial^2 T}{\partial z^2} \right) + \frac{S}{\rho \hat{C}_p}$$

Cylindrical (r $\theta$ z) coordinates:

$$\frac{\partial T}{\partial t} + v_r \frac{\partial T}{\partial r} + \frac{v_\theta}{r} \frac{\partial T}{\partial \theta} + v_z \frac{\partial T}{\partial z} = \frac{k}{\rho \hat{C}_p} \left( \frac{1}{r} \frac{\partial}{\partial r} \left( r \frac{\partial T}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 T}{\partial \theta^2} + \frac{\partial^2 T}{\partial z^2} \right) + \frac{S}{\rho \hat{C}_p}$$

Spherical (r $\theta$ ) coordinates:

$$\frac{\partial T}{\partial t} + v_r \frac{\partial T}{\partial r} + \frac{v_\theta}{r} \frac{\partial T}{\partial \theta} + \frac{v_\phi}{r \sin \theta} \frac{\partial T}{\partial \phi} = \frac{k}{\rho \hat{C}_p} \left( \frac{1}{r^2} \frac{\partial}{\partial r} \left( r^2 \frac{\partial T}{\partial r} \right) + \frac{1}{r^2 \sin \theta} \frac{\partial}{\partial \theta} \left( \sin \theta \frac{\partial T}{\partial \theta} \right) + \frac{1}{r^2 \sin^2 \theta} \frac{\partial^2 T}{\partial \phi^2} \right) + \frac{S}{\rho \hat{C}_p}$$

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1D Heat Transfer: Unsteady State

Microscopic Energy Equation in Cartesian Coordinates

$$\frac{\partial T}{\partial t} + v_x \frac{\partial T}{\partial x} + v_y \frac{\partial T}{\partial y} + v_z \frac{\partial T}{\partial z} = \frac{k}{\rho \hat{C}_p} \left( \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} + \frac{\partial^2 T}{\partial z^2} \right) + \frac{S}{\rho \hat{C}_p}$$

$$\alpha \equiv \frac{k}{\rho \hat{C}_p} = \text{thermal diffusivity}$$

what are the boundary conditions? initial conditions?

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1D Unsteady Heat Transfer: Moderate and High Biot Number

## Boundary Conditions:

- Nonzero resistance to heat transfer:  $\left| \frac{q_x}{A} \right| = |h(T - T_b)|$
- No resistance to heat transfer:  $h \rightarrow \infty$ , or, equivalently, temperature known at the boundary
- Insulated boundary:  $\left| \frac{q_x}{A} \right| = 0$ , or, equivalently, the temperature field is symmetrical at the boundary

1D Heat Transfer: Unsteady State

Microscopic Energy Equation in Cartesian Coordinates

$$\frac{\partial T}{\partial t} + v_x \frac{\partial T}{\partial x} + v_y \frac{\partial T}{\partial y} + v_z \frac{\partial T}{\partial z} = \frac{k}{\rho \hat{C}_p} \left( \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} + \frac{\partial^2 T}{\partial z^2} \right) + \frac{S}{\rho \hat{C}_p}$$

$\alpha \equiv \frac{k}{\rho \hat{C}_p}$

thermal diffusivity

what are the boundary conditions? initial conditions?

## Initial Conditions:

- Initial temperature distribution uniform:  $T_0$
- Initial temperature distribution known:  $T(o, \underline{x})$

13  
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1D Unsteady Heat Transfer: Moderate and High Biot Number

For the most common geometries, initial conditions, and boundary conditions, the models have been solved and may be looked-up.

For quick, “back of the envelope” calculations, researchers (Gurney and Lurie, Heisler) have created easy-to-use plots of the predictions.

Conduction of Heat in Solids

SECOND EDITION

OXFORD AT THE CLARENDON PRESS

H. S. CARSLAW and J. C. JAEGER

1947

14  
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1D Unsteady Heat Transfer: Finite Bodies, Short Cut Solutions

**Finite 1D Unsteady Heat Transfer,  $T = T(t, x)$  or  $T = T(t, r)$**

**Initial:** Uniform initial temperature  $T_0$ ; **BC:** exposed to bulk temperature  $T_1$ ;  $h$  known

- Flat plate long, wide, thickness =  $2x_1$      $T = T(t, x)$      $Y = Y(X, n)$
- Cylinder long, radius =  $x_1$      $T = T(t, r)$      $Y = Y(X, n)$
- Sphere radius =  $x_1$      $T = T(t, r)$      $Y = Y(X, n)$

**Note:**  
 $D_{char} = x_1$ ,  
NOT  $V/A$

**Heisler Charts ( $T(Fo, 0)$ ) and Gurney-Lurie Charts ( $T(Fo, n)$ )** are graphical representations of solutions of a particular unsteady heat-transfer problem for various values of  $m = 1/Bi$ .

$$Bi = \frac{hD_{char}}{k} = \frac{hx_1}{k} = \frac{1}{m}$$

$$Fo = \frac{\alpha t}{x_1^2} = X$$

$$\frac{x}{x_1} = \frac{r}{x_1} = n$$

$$\frac{T_1 - T}{T_1 - T_0} = Y \quad \left( \frac{T - T_0}{T_1 - T_0} = 1 - Y \right)$$

1D Unsteady Heat Transfer: Finite Bodies

**Gurney and Lurie Charts**

Ref: Geankoplis, 4<sup>th</sup> Ed, 2003

**Initial:** Uniform initial temperature  $T_0$ ; **BC:** bulk temperature  $T_1$ ;  $T \left( m = \frac{1}{Bi}, n = \frac{x}{x_1}, Fo \right)$

- Flat plate long, wide, thickness =  $2x_1$      $T = T(t, x)$      $Y = Y(X, n)$
- Cylinder long, radius =  $x_1$      $T = T(t, r)$      $Y = Y(X, n)$
- Sphere radius =  $x_1$      $T = T(t, r)$      $Y = Y(X, n)$

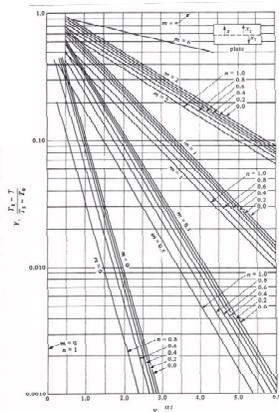


FIGURE 5.3-5 Unsteady-state heat conduction in a large flat plate. (From H. P. Gurney and J. Lurie, Ind. Eng. Chem., 38, 1170 (1925).)

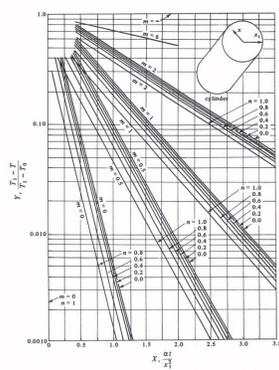


FIGURE 5.3-7 Unsteady-state heat conduction in a long cylinder. (From H. P. Gurney and J. Lurie, Ind. Eng. Chem., 38, 1170 (1925).)

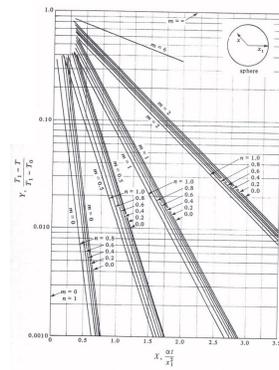


FIGURE 5.3-9 Unsteady-state heat conduction in a sphere. (From H. P. Gurney and J. Lurie, Ind. Eng. Chem., 38, 1170 (1925).)

$$Fo = \frac{\alpha t}{x_1^2} = X$$

1D Unsteady Heat Transfer: Finite Bodies

**Heisler Charts**

**Initial:** Uniform initial temperature  $T_0$  **Flat plate**  
**BC:** Exposed to bulk temperature  $T_1$   
 $h$  known  
 Plots of temperature at the center

$T \left( m = \frac{1}{Bi}, x = 0, Fo \right)$

**Cylinder**

**Sphere**

Ref: Geankoplis, 4<sup>th</sup> Ed, 2003

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$Fo = \frac{\alpha t}{x_1^2} = X$  17

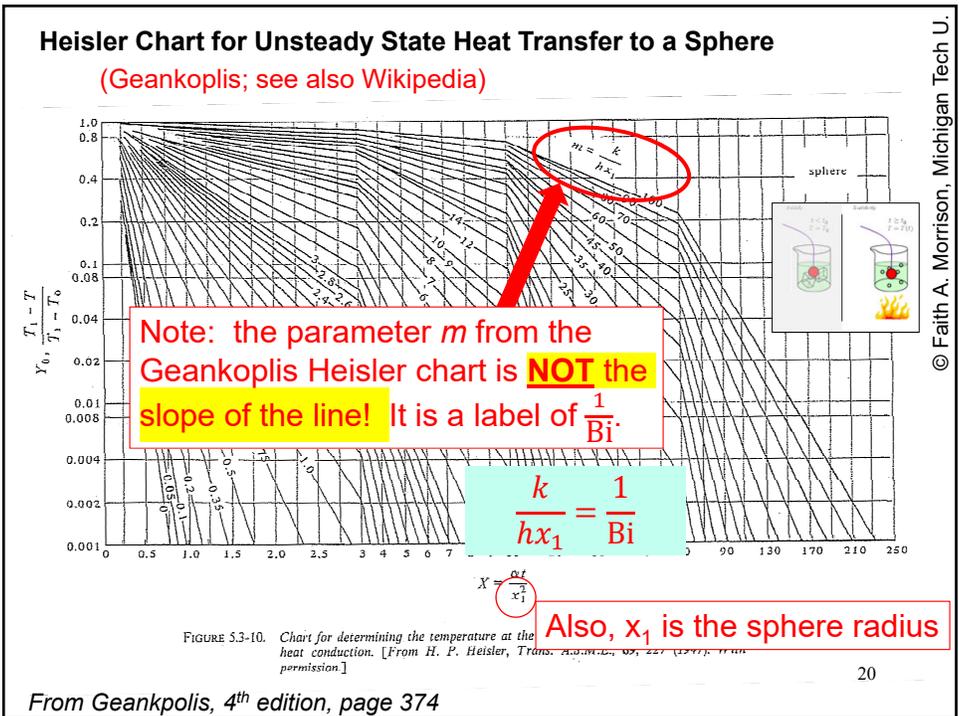
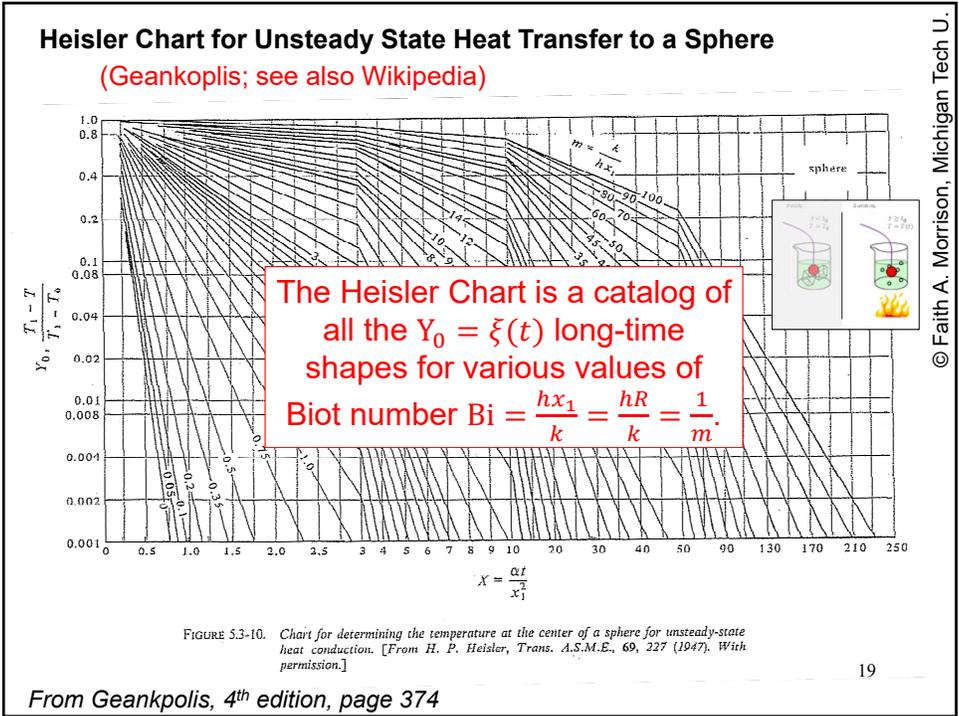
**Heisler Chart for Unsteady State Heat Transfer to a Sphere**  
 (Geankoplis; see also Wikipedia)

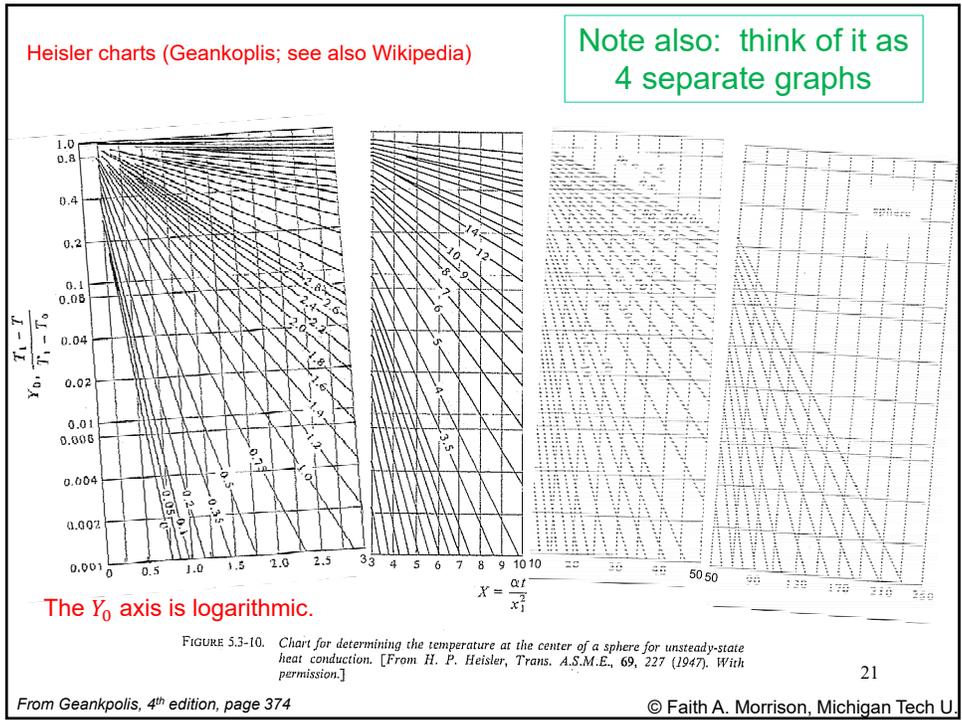
FIGURE 5.3-10. Chart for determining the temperature at the center of a sphere for unsteady-state heat conduction. [From H. P. Heisler, Trans. A.S.M.E., 69, 227 (1947). With permission.]

From Geankoplis, 4<sup>th</sup> edition, page 374

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18





1D Unsteady Heat Transfer: In a Slab

**Example:** A tall, wide rectangular copper 304 stainless steel slab, five ten centimeters thick, uniformly at a temperature of  $17^\circ\text{C}$ , is suddenly exposed on all sides to air water ( $h = 1380 \text{ W/m}^2\text{K}$ ) at  $45^\circ\text{C}$ . After 30 20 minutes, what is the temperature at the middle of the slab?

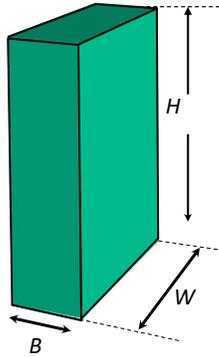
(numbers were changed in 2021 to improve the problem; the old numbers appear in the video)

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1D Unsteady Heat Transfer: In a Slab

**Example:** A tall, wide rectangular 304 stainless steel slab, ten centimeters thick, uniformly at a temperature of  $17^{\circ}C$ , is suddenly exposed on all sides to water ( $h = 1380 W/m^2K$ ) at  $45^{\circ}C$ . After 20 minutes, what is the temperature at the middle of the slab?

Let's  
try



23

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1D Unsteady Heat Transfer: In a Slab

**Example:** A long, wide rectangular slab of butter (46 mm thick) at  $4.4^{\circ}C$  is removed from refrigeration and placed on a table at room temperature. After five hours, what are the butter temperatures at the middle of the slab and at the bottom of the slab (in contact with the table)?

Properties of butter:

$$k_{butter} = 0.197 \frac{W}{mK}$$

$$\hat{C}_p_{butter} = 2.30 \frac{kJ}{kg K}$$

$$\rho_{butter} = 998 \frac{kg}{m^3}$$

Conditions of the room:

$$T_{bulk} = 24^{\circ}C$$

$$h_{conv} = 8.5 \frac{W}{m^2K}$$

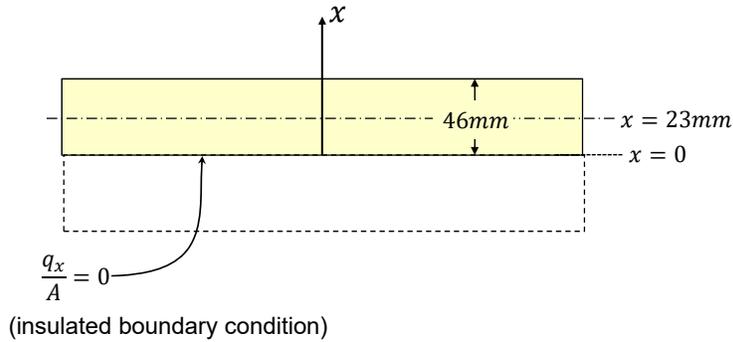
Handnotes: [https://pages.mtu.edu/~fmorriso/cm3120/2021\\_solve\\_Gurney-Lurie\\_problem\\_butter.pdf](https://pages.mtu.edu/~fmorriso/cm3120/2021_solve_Gurney-Lurie_problem_butter.pdf)

24

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1D Unsteady Heat Transfer: In a Slab

**Example:** A long, wide rectangular slab of butter (46 mm thick) at 4.4°C is removed from refrigeration and placed on a table at room temperature. After five hours, what are the butter temperatures at the middle of the slab and at the bottom of the slab (in contact with the table)?



25

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The Y axis is logarithmic.

$Fo = \frac{\alpha t}{x_1^2} = X$

### Gurney and Lurie Chart Finite 1D Unsteady Heat Transfer

**Initial:** Uniform initial temperature  $T_0$

**BC:** Exposed to bulk temperature  $T_1$

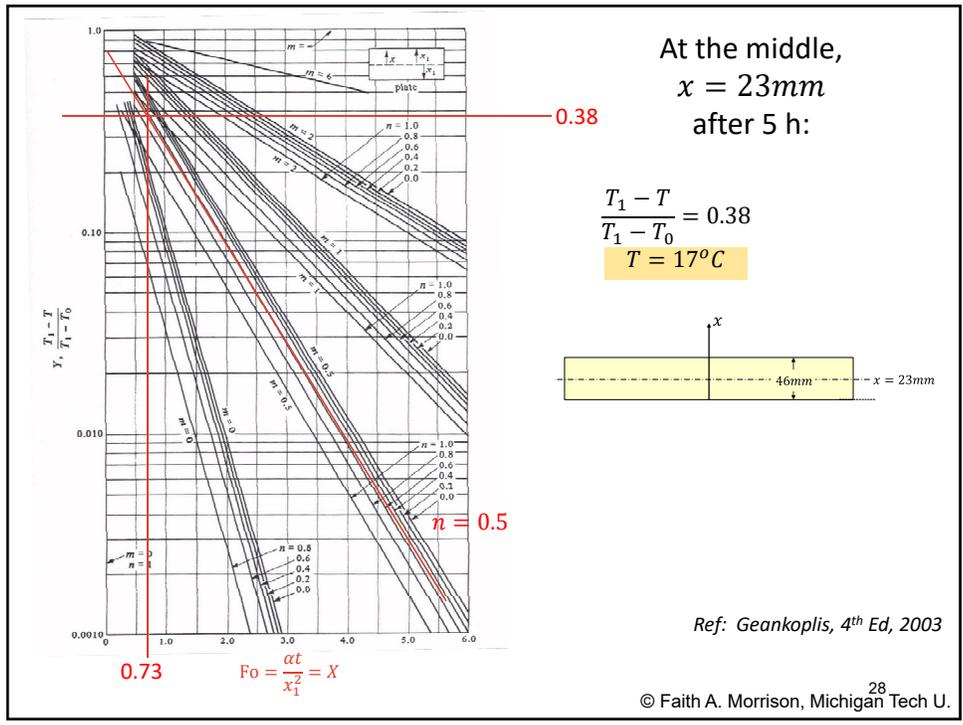
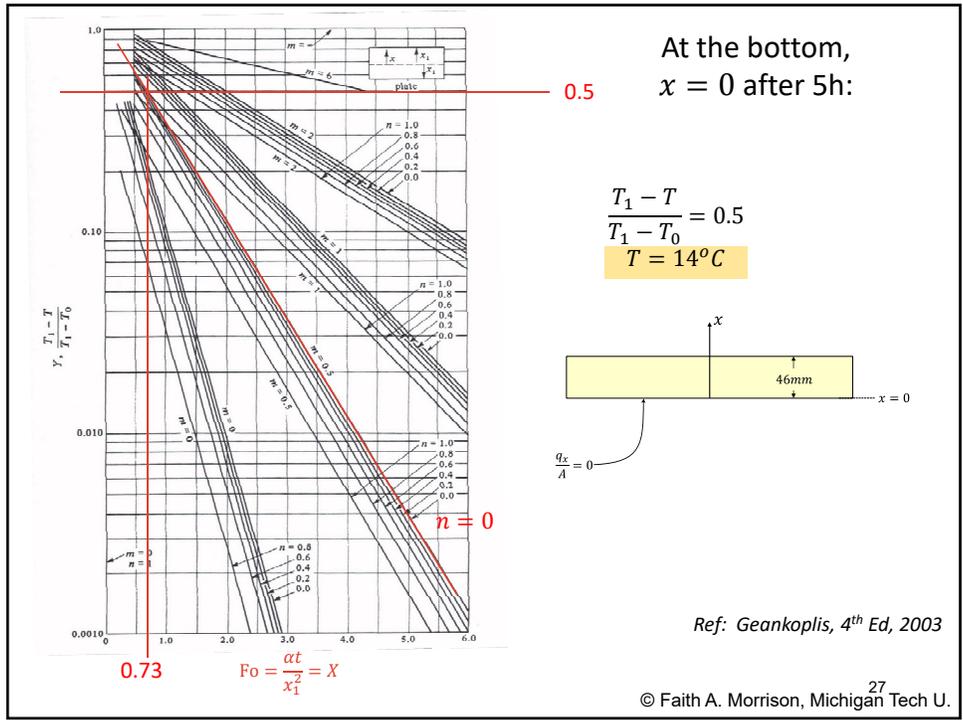
$h$  known

- Flat plate, long, wide
- thickness  $2x_1$
- $T = T(t, x)$
- $Y = Y\left(Fo, n = \frac{x}{x_1}\right)$

$T\left(m = \frac{1}{Bi}, n = \frac{x}{x_1}, Fo\right)$

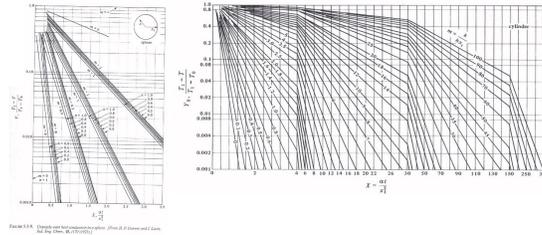
*Ref: Geankoplis, 4<sup>th</sup> Ed, 2003*

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1D Unsteady Heat Transfer: Moderate and High Biot Number

Summary of low Moderate/High Biot number scenarios (short-cut solutions):



- Good for quick “back of the envelope” calculations
- Uniform starting temperature
- If surface temperature is known,  $Bi = \infty$ , that is,  $m = 0$
- If boundary is insulated (flat plate), set that boundary as the center of the plate ( $q_x/A = 0$ ) boundary condition)
- Limited sig figs (*i.e.* 1-2, due to reading charts, interpolating)

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Next: Complete solutions for Moderate and High Biot number behavior

We indicated that there are three ranges of Biot number to consider:

$$Bi = \frac{D_{char}/k}{1/h}$$

At high Bi, the surface temperature equals the bulk temperature; heat transfer is limited by conduction in the body.

**High Bi:**  
low  $k$ ,  
high  $h$

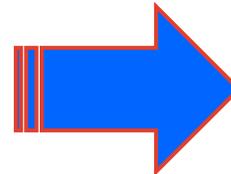
At moderate Bi, heat transfer is affected by both conduction in the body and the rate of heat transfer to the surface.

**Moderate Bi:**  
nether process dominates

At low Bi, the temperature is uniform in a finite body; heat transfer is limited by rate of heat transfer to the surface ( $h$ ).

**Low Bi:**  
high  $k$ ,  
low  $h$

We now explore these ranges



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