CM3120 Transport/Unit Operations 2

Unsteady State Heat Transfer





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www.chem.mtu.edu/~fmorriso/cm3120/cm3120.html

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Where to start?

We seek to study unsteady state heat transfer.

Let's start by looking over several subjects that form the foundation for what we hope to study.

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These basic concepts are familiar. Cycling back can deepen our understanding (and help us put new concepts in context)

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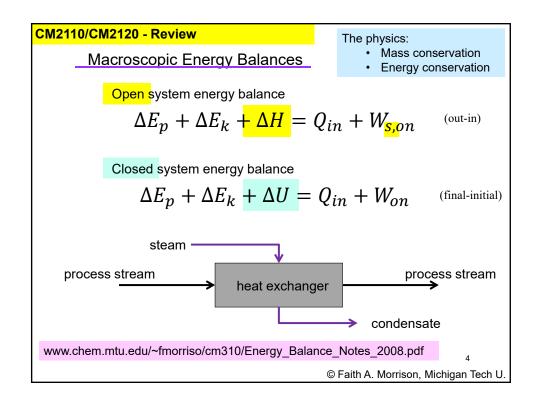
Energy Balance Review





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CM2110/CM2120 - Review

Review:

How do we decide what equations to use for what?

- · Closed system E-bal (first law of thermo)
- Open system E-bal (H = U +PV, flowing systems)
- Mechanical energy balance (SISO, steady, isothermal, no rxn, no phase change, little heat transferred)

Knowing what assumptions we are making means we understand our models. See handout for summary: Macroscopic Energy Balances Open system energy balance $\Delta E_p + \Delta E_k + \Delta H = Q_{in} + W_{s,on}$ Closed system energy balance $\Delta E_p + \Delta E_k + \Delta U = Q_{in} + W_{on}$ MEB: $\frac{\Delta p}{\rho} + \frac{\Delta(v^2)}{2\alpha} + g\Delta z + F = \frac{W_{s,on}}{\dot{m}}$

Notes:

- 1. Δ has different meanings in the three e-balances

$$\Delta H = \sum_{outs} m_i \widehat{H}_i - \sum_{ins} m_i \widehat{H}_i$$
 3. Use MEB if you can (easy); but

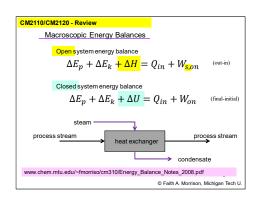
not if it does not appy!

www.chem.mtu.edu/~fmorriso/cm310/Energy Balance Notes 2008.pdf

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What *physics*

determines how rapidly the heat transfers from the outside stream to the inside stream?



Fourier's Law of Heat Conduction

$$\frac{q_x}{A} = -k \frac{dT}{dx}$$

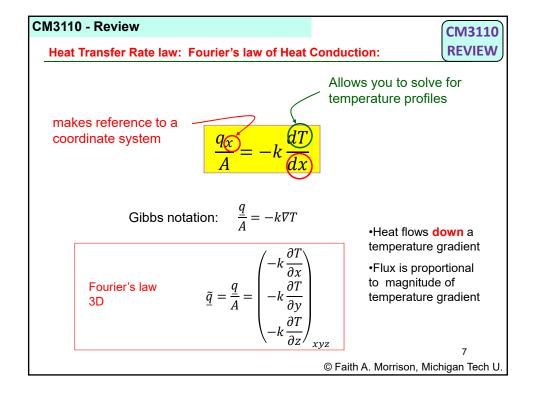
(for a homogeneous phase)

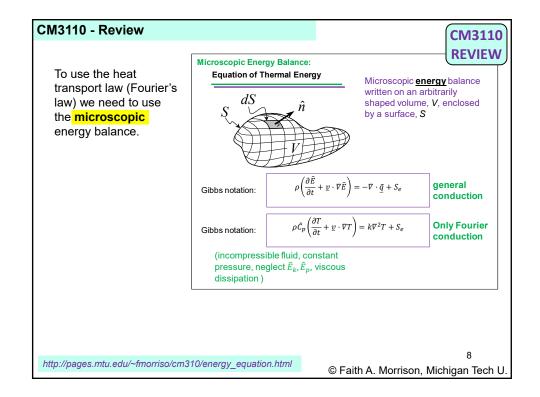
 $\frac{q_x}{4}$ -heat flux=energy/area time)

 $\frac{1}{k}$ - thermal conductivity

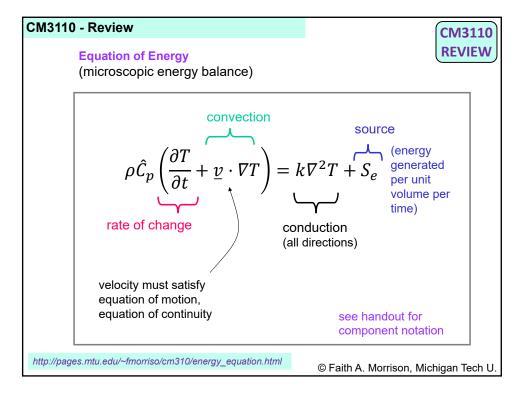
 $\frac{dT}{dx}$ –temperature gradient

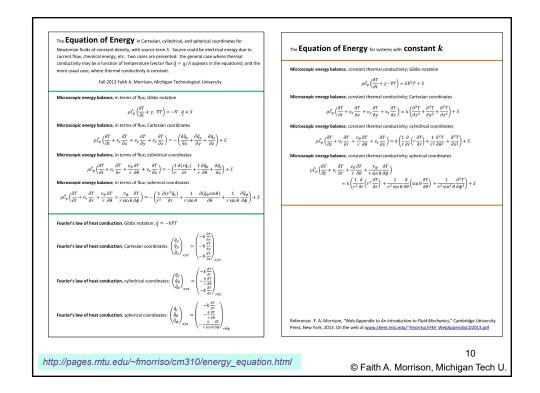
(the driving physics is Brownian motion: energy transports down ∇T due to Brownian motion)

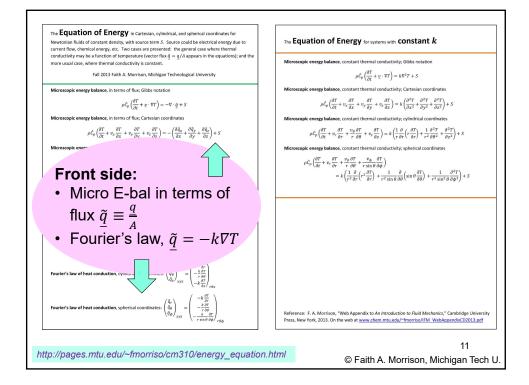


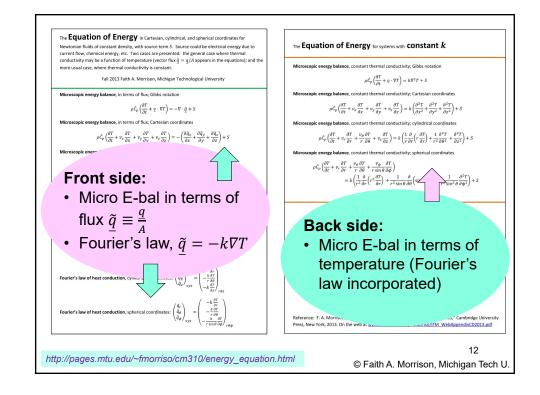


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Microscopic Energy Balance

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The Equation of Energy for systems with constant $m{k}$

Microscopic energy balance, constant thermal conductivity; Gibbs notation

$$\rho \hat{C}_p \left(\frac{\partial T}{\partial t} + \underline{v} \cdot \nabla T \right) = k \nabla^2 T + S$$

Microscopic energy balance, constant thermal conductivity; Cartesian coordinates

$$\rho \hat{C}_p \left(\frac{\partial T}{\partial t} + v_x \frac{\partial T}{\partial x} + v_y \frac{\partial T}{\partial y} + v_z \frac{\partial T}{\partial z} \right) = k \left(\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} + \frac{\partial^2 T}{\partial z^2} \right) + S$$

Microscopic energy balance, constant thermal conductivity; cylindrical coordinates

$$\rho \hat{C}_p \left(\frac{\partial T}{\partial t} + v_r \frac{\partial T}{\partial r} + \frac{v_\theta}{r} \frac{\partial T}{\partial \theta} + v_z \frac{\partial T}{\partial z} \right) = k \left(\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial T}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 T}{\partial \theta^2} + \frac{\partial^2 T}{\partial z^2} \right) + S$$

Microscopic energy balance, constant thermal conductivity; spherical coordinates

$$\begin{split} \rho \hat{\mathcal{C}}_p \left(\frac{\partial T}{\partial t} + v_r \frac{\partial T}{\partial r} + \frac{v_\theta}{r} \frac{\partial T}{\partial \theta} + \frac{v_\phi}{r \sin \theta} \frac{\partial T}{\partial \phi} \right) \\ &= k \left(\frac{1}{r^2} \frac{\partial}{\partial r} \left(r^2 \frac{\partial T}{\partial r} \right) + \frac{1}{r^2 \sin \theta} \frac{\partial}{\partial \theta} \left(\sin \theta \frac{\partial T}{\partial \theta} \right) + \frac{1}{r^2 \sin^2 \theta} \frac{\partial^2 T}{\partial \phi^2} \right) + \mathcal{S} \end{split}$$

http://pages.mtu.edu/~fmorriso/cm310/energy_equation.html

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Fourier's Law of Heat Conduction

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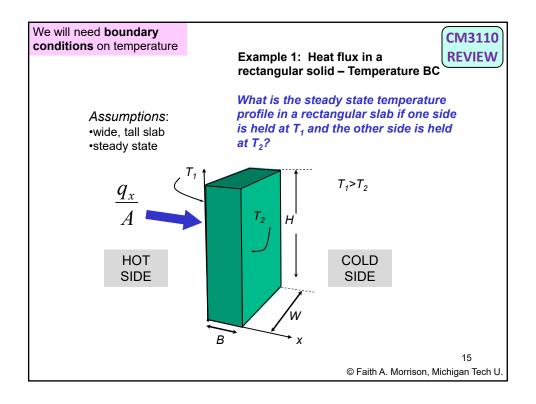
Fourier's law of heat conduction, Gibbs notation: $\tilde{q} = -k\nabla T$

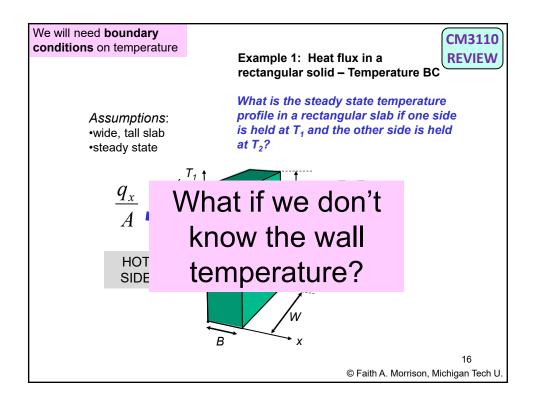
Fourier's law of heat conduction, Cartesian coordinates:
$$\begin{pmatrix} \tilde{q}_x \\ \tilde{q}_y \\ \tilde{q}_z \end{pmatrix}_{xyz} = \begin{pmatrix} -k \frac{\partial T}{\partial x} \\ -k \frac{\partial T}{\partial y} \\ -k \frac{\partial T}{\partial z} \end{pmatrix}_{xy}$$

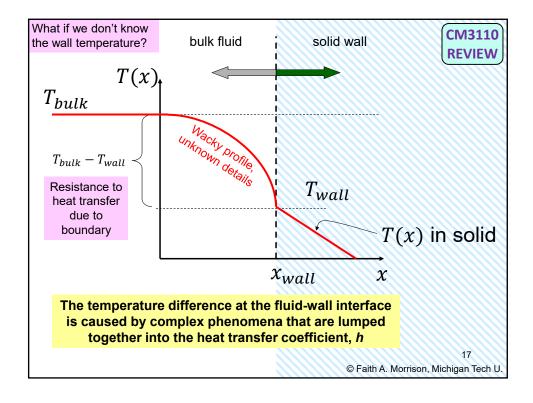
Fourier's law of heat conduction, cylindrical coordinates:
$$\begin{pmatrix} \tilde{q}_r \\ \tilde{q}_\theta \\ \tilde{q}_z \end{pmatrix}_{xyz} = \begin{pmatrix} -k \frac{\partial T}{\partial r} \\ -\frac{k}{r} \frac{\partial T}{\partial \theta} \\ -k \frac{\partial T}{\partial z} \end{pmatrix}_{r\theta z}$$

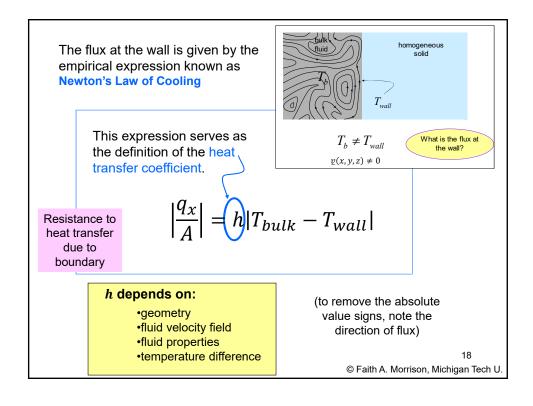
Fourier's law of heat conduction, spherical coordinates:
$$\begin{pmatrix} \tilde{q}_r \\ \tilde{q}_\theta \\ \tilde{q}_\phi \end{pmatrix}_{xyz} = \begin{pmatrix} -k \frac{\partial T}{\partial r} \\ -\frac{k}{r} \frac{\partial T}{\partial \theta} \\ -\frac{k}{r \sin \theta} \frac{\partial T}{\partial \phi} \end{pmatrix}_{r\theta\phi}$$

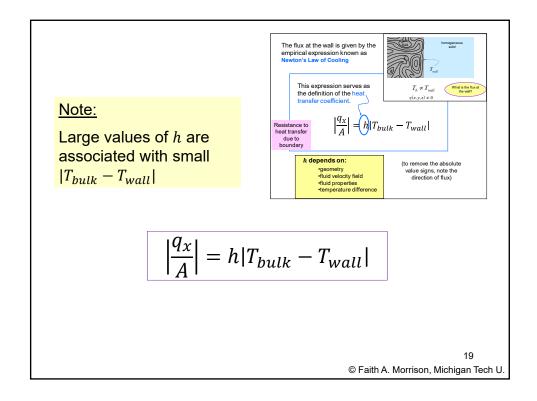
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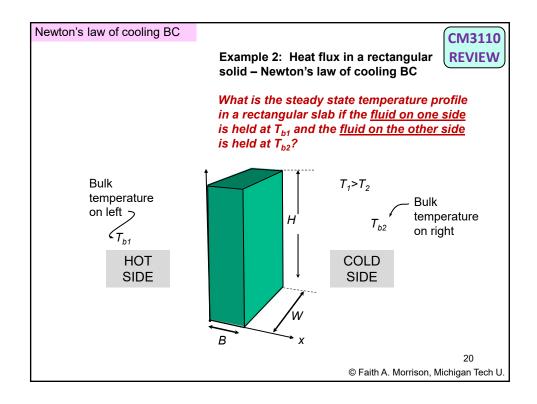


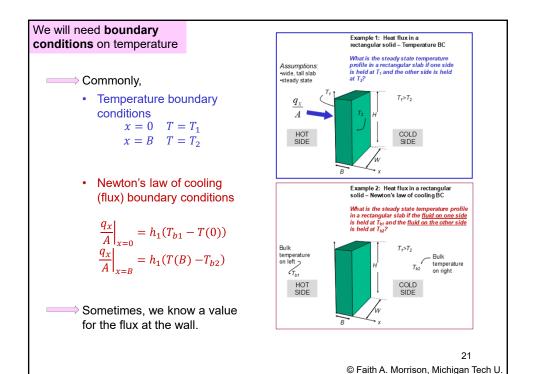


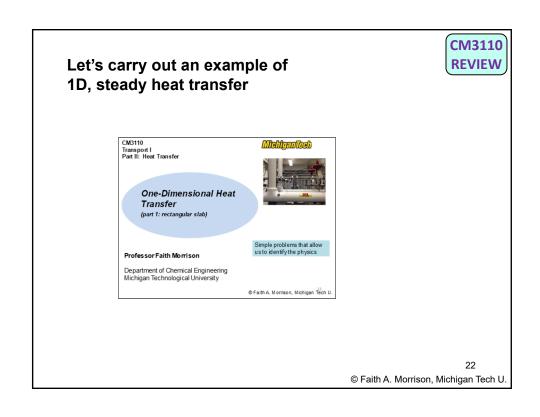




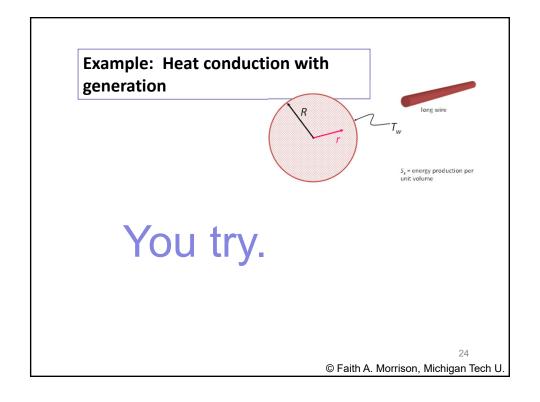


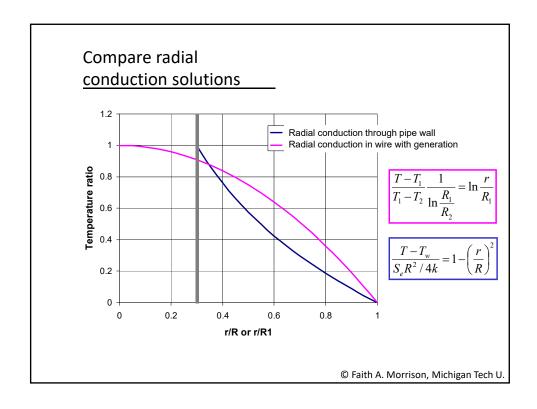


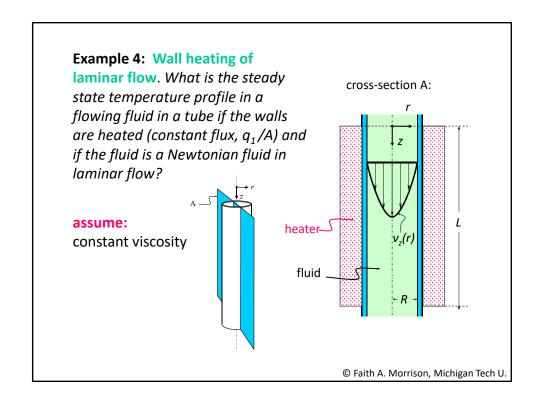


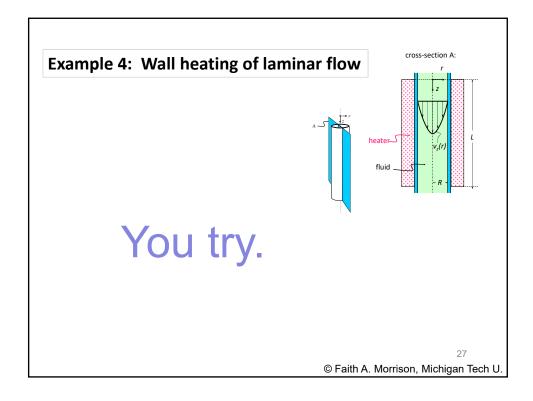


Example 3: Heat Conduction with Generation What is the steady state temperature profile in a wire if heat is generated uniformly throughout the wire at a rate of S_e W/m³ and the outer radius is held at T_w ? What is the flux? $S_e = \text{energy}$ production per unit volume

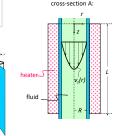








Example 4: Wall heating of laminar flow



We need to solve this partial differential equation:

$$\frac{1}{r}\frac{\partial}{\partial r}\left(\frac{\partial T}{\partial r}r\right) + \frac{\partial}{\partial z}\left(\frac{\partial T}{\partial z}\right) - \frac{\rho\hat{C}_p}{k}v_z(r)\frac{\partial T}{\partial z} = 0$$

with

$$v_z = \frac{\Delta p}{4\mu L} (R^2 - r^2)$$

and with the appropriate boundary conditions. To see the solution go to:

- R. Siegel, E. M. Sparrow, T. M. Hallman, Appl. Science Research A7, 386-392 (1958)
- R. B. Bird, W. Stewart, and E. Lightfoot (BSL), Transport Phenomena, Wiley, 1960, p295.

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We need to solve this partial differential equation:

$$\frac{1}{r}\frac{\partial}{\partial r} \bigg(\frac{\partial T}{\partial r} r \bigg) + \frac{\partial}{\partial z} \bigg(\frac{\partial T}{\partial z} \bigg) - \frac{\rho \hat{\mathcal{C}}_p}{k} \, v_z(r) \frac{\partial T}{\partial z} = 0$$

with

$$v_z = \frac{\Delta p}{4\mu L} (R^2 - r^2)$$

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Example 4: Wall heating of laminar flow

$$r = 0$$
 $T = finite$
 $r = R$ $\frac{q_r}{A} = \frac{q_1}{A}$
 $z = 0$ $T = T_0$

1D, steady heat transfer

CM3110 REVIEW

SUMMARY

- · Microscopic energy balance
- Transport law
- Newton's law of cooling (fluid boundary)
- Know the assumptions that simplify the model of the problem
- · Solve with appropriate boundary conditions
- Check that assumptions are valid when using the solution

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