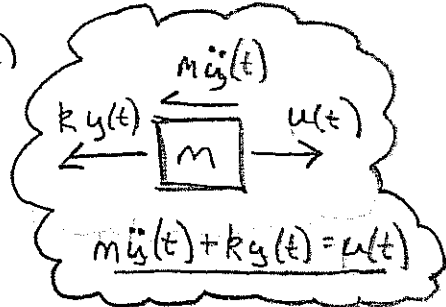
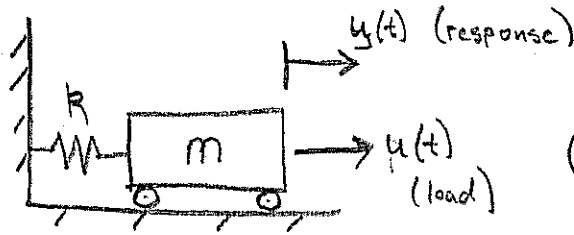


LESSON 5 - FREQ. DOMAIN REPRESENTATION OF SYSTEMS:

- OBJECTIVES: • DEFINE IMPULSE RESPONSE
• CONVOLUTION IN F.D.

1) IMPULSE RESPONSE:

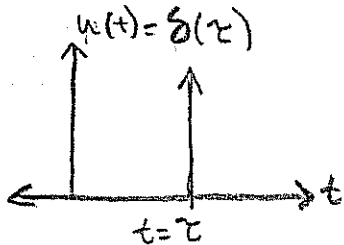
→ CONSIDER A MASS-SPRING SYSTEM:



Make some assumptions:

- 1.) Linear system
- 2.) Time-invariant system
- 3.) zero initial conditions $\begin{cases} y(0) = 0 \\ \dot{y}(0) = 0 \end{cases}$

• Let $u(t)$ be an impulse load:

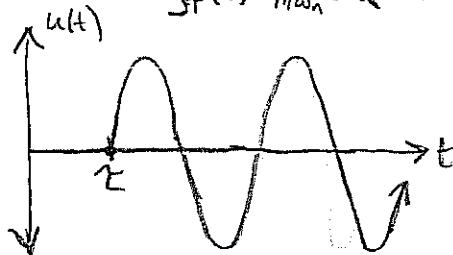


Define $\omega_n = \sqrt{\frac{k}{m}}$ (Natural Frequency)

$$m\ddot{y}(t) + ky(t) = \delta(\tau)$$

→ Solution:

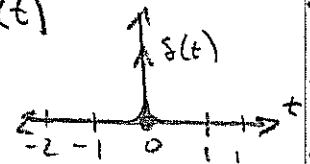
- $t < \tau \rightarrow y_p(t) = 0$
- $t \geq \tau \rightarrow y_p(t) = \frac{1}{m\omega_n} \sin(\omega_n(t-\tau))$



DEFINITION:

• DELTA-DIRAC FUNCTION

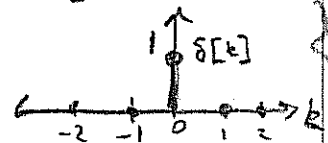
CT: $\delta(t)$



$$\int_{-\infty}^{\infty} \delta(t) dt = 1$$

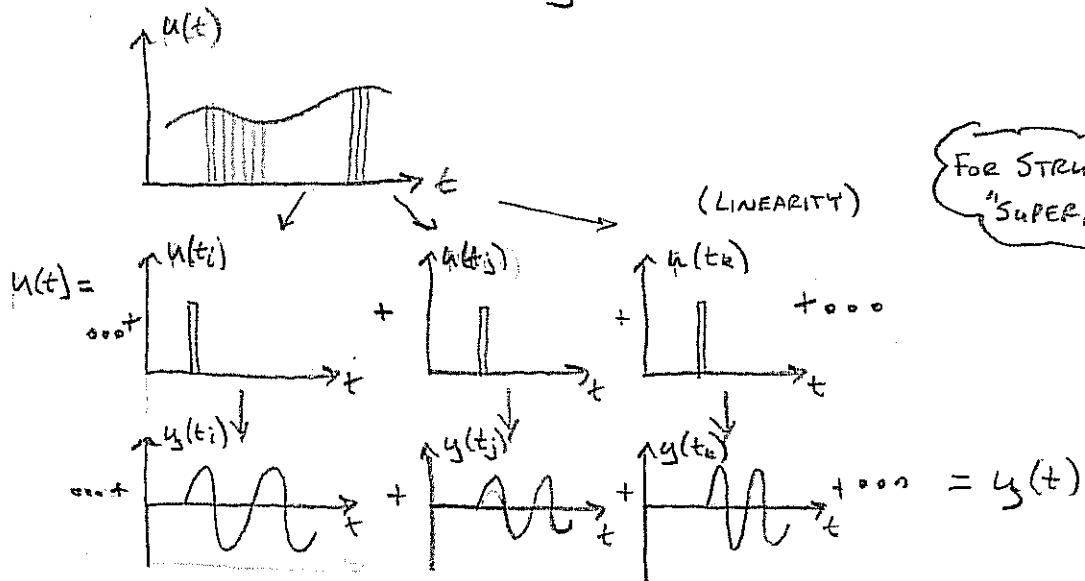
• KRONECKER DELTA

DT: $\delta[k]$



(UNIT IMPULSE FUNCTIONS)
IN CT & DT

• LET $u(t)$ be an arbitrary load:



• IDEA BEHIND CONVOLUTION INTEGRAL:

$$y(t) = \int_{-\infty}^{\infty} u(\tau) h(t-\tau) d\tau$$

OR: $\underline{u(t) \otimes h(t)}$

$h(t)$ = impulse response of the system:

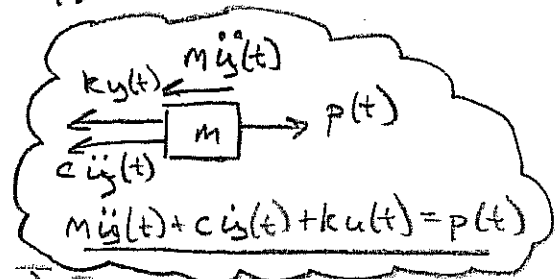
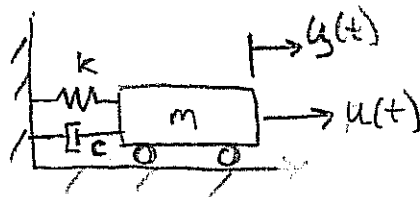
• FOR MASS-SPRING SYSTEM:

$$y(t) = \int_{-\infty}^{\infty} \frac{u(\tau)}{m \omega_n} \underbrace{\sin \omega_n (t-\tau)}_{h(t-\tau)} d\tau$$

$$h(t) = \frac{\sin \omega_n t}{m \omega_n}$$

→ MOST REAL-WORLD SYSTEMS HAVE SOME FORM OF DAMPING:

E.G. MASS-SPRING-DASHPOT SYSTEM



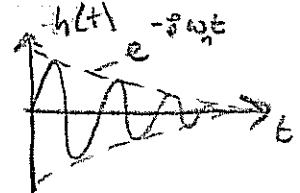
$$y(t) = \int_{-\infty}^{\infty} \frac{u(\tau)}{m\omega_D} e^{-\xi\omega_n(t-\tau)} \sin \omega_D(t-\tau) d\tau$$

↑ $h(t-\tau)$

$$h(t) = \frac{e^{-\xi\omega_n t} \sin \omega_D t}{m\omega_D}$$

$$\omega_D = \omega_n \sqrt{1 - \xi^2}$$

$$\xi = \frac{c}{2m\omega_n}$$



NOTE: DECAYING NATURE!
INDICATES BIBO STABILITY.

• CONVOLUTION INTEGRAL CAN PROVIDE CLOSED-FORM RESPONSE TO ARBITRARY INPUT...

→ BUT, MAY BE VERY DIFFICULT TO SOLVE'S

- CONSIDER NUMERICAL ANALYSIS

-OR-

- SOLVE IN FREQUENCY DOMAIN

2) FOURIER TRANSFORM OF CONVOLUTION INTEGRAL

CONVOLUTION INTEGRAL: $y(t) = u(t) \otimes h(t)$

$$y(t) = \int_{-\infty}^{\infty} u(\tau) h(t-\tau) d\tau$$

$$\Downarrow \mathcal{F}$$

$$\mathcal{F}(y(t) = \int_{-\infty}^{\infty} u(\tau) h(t-\tau) d\tau)$$

$$Y(\omega) = \mathcal{F}\left(\int_{-\infty}^{\infty} u(\tau) h(t-\tau) d\tau\right)$$

$$Y(\omega) = \int_{-\infty}^{\infty} \frac{1}{2\pi} \left(\int_{-\infty}^{\infty} u(\tau) h(t-\tau) d\tau \right) e^{-i\omega t} dt$$

$$Y(\omega) = \int_{-\infty}^{\infty} \frac{1}{2\pi} \int_{-\infty}^{\infty} h(t-\tau) e^{-i\omega t} dt u(\tau) d\tau$$

define: $\alpha = t - \tau$
 $d\alpha = dt$

$$Y(\omega) = \int_{-\infty}^{\infty} \frac{1}{2\pi} \int_{-\infty}^{\infty} h(\alpha) e^{-i\omega \alpha} d\alpha e^{-i\omega \tau} u(\tau) d\tau$$

$$Y(\omega) = \int_{-\infty}^{\infty} \underbrace{H(\omega)}_{\text{this is } H(\omega) \leftarrow \text{a.k.a. "Transfer Function"}} e^{-i\omega \tau} u(\tau) d\tau$$

$$Y(\omega) = 2\pi H(\omega) \cdot \underbrace{\frac{1}{2\pi} \int_{-\infty}^{\infty} u(\tau) e^{-i\omega \tau} d\tau}_{\text{this is } U(\omega)}$$

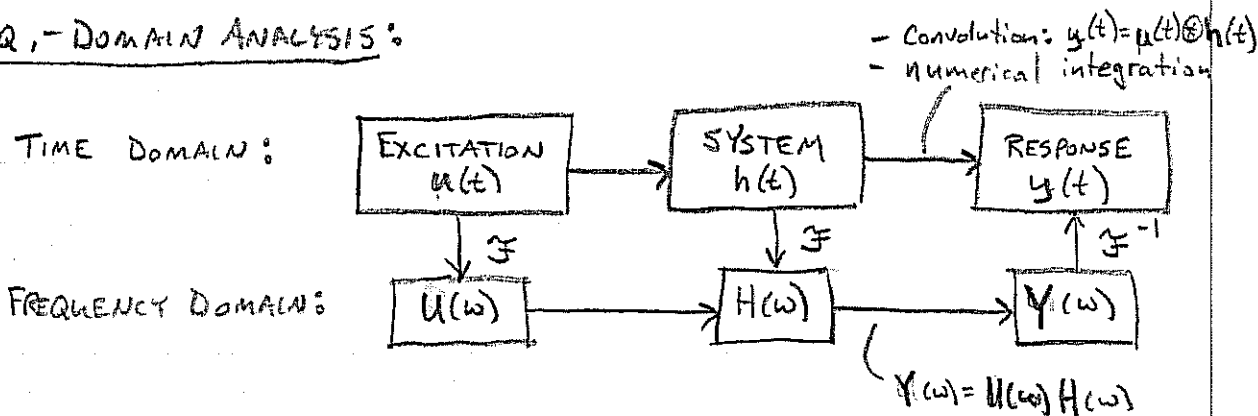
$$Y(\omega) = 2\pi H(\omega) U(\omega)$$

- CONVOLUTION BECOMES SIMPLE MULTIPLICATION IN FREQUENCY DOMAIN!

Remember the frequency-domain filters?

- Anti-aliasing filters are multiplied in freq.-domain and convolved in time-domain.

- Time-domain windows are multiplied in the time-domain and convolved in the freq.-domain!

FREQ. - DOMAIN ANALYSIS:

→ If $u(t)$ is replaced with probability density function of input statistics you get statistics of output

- RANDOM VIBRATION THEORY
- STOCHASTIC PROCESSES

ADVANTAGES:

- ① CONVOLUTION BECOMES SIMPLE MULTIPLICATION
 - ② FREQUENCY-DOMAIN THINKING USEFUL IN ENGINEERING
 - ③ PROPERTIES OF $H(\omega)$ CAN SAY A LOT ABOUT SYSTEM
 - ④ CAN DETERMINE $H(\omega)$ EXPERIMENTALLY FROM DATA
- SYSTEM ID

DISADVANTAGES:

- ① REQUIRES LINEAR, TIME-INVARIANT ASSUMPTION TO BE VALID
- ② INVERSE FOURIER INTEGRAL CAN BE TROUBLESOME

→ We will focus on discrete-time representations of systems over limited time horizons:
- This focus will simplify our analyses greatly,

→ Our System ID approaches will yield linearized models of (potentially) non-linear systems,